



Research article

Interaction of rigid body motion and rarefied gas dynamics based on the BGK model[†]

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Abstract: In this paper we present simulations of moving rigid bodies immersed in a rarefied gas. The rarefied gas is simulated by solving the Bhatnager-Gross-Krook (BGK) model for the Boltzmann equation. The Newton-Euler equations are solved to simulate the rigid body motion. The force and the torque on the rigid body is computed from the surrounded gas. An explicit Euler scheme is used for the time integration of the Newton-Euler equations. The BGK model is solved by the semi-Lagrangian method suggested by Russo & Filbet [22]. Due to the motion of the rigid body, the computational domain for the rarefied gas (and the interface between the rigid body and the gas domain) changes with respect to time. To allow a simpler handling of the interface motion we have used a meshfree method for the interpolation procedure in the semi-Lagrangian scheme. We have considered a one way, as well as a two-way coupling of rigid body and gas flow. We use diffuse reflection boundary conditions on the rigid body and also on the boundary of the computational domain. In one space dimension the numerical results are compared with analytical as well as with Direct Simulation Monte Carlo (DSMC) solutions of the Boltzmann equation. In the two-dimensional case results are compared with DSMC simulations for the Boltzmann equation and with results obtained by other researchers. Several test problems and applications illustrate the versatility of the approach.

Keywords: rigid body motion; rarefied gas; kinetic equation; BGK model; meshfree method; semi-Lagrangian method

1. Introduction

In recent years moving boundary problems for rarefied gas dynamics have been extensively investigated in the connection with Micro-Electro-Mechanical-Systems (MEMS), see [4, 10, 11, 16, 22, 24, 25, 29–31]. In micro scale geometries the mean free path is often of the order or larger than the characteristic length of the geometry, requiring the solution of kinetic equations. Usually, these flows have low Mach numbers, therefore, stochastic methods like DSMC are not the optimal choice, since statistical noise dominates the flow quantities. Moreover, when one considers moving rigid body, the gas domain will change in time and one has to encounter unsteady flow problems such that averages over long runs cannot be taken. Instead, one has to perform many independent runs in order to get smooth solutions. Although some attempts have been made to reduce the statistical noise, see, for example, [12], many works rather employ deterministic approaches for simplified models of the Boltzmann equation, like the Bhatnagar-Gross-Krook (BGK) model, see [10, 22, 30].

In this paper we are following deterministic approach to solve the BGK model and extend the semi-Lagrangian method suggested in [22] to two dimensions in physical space and three dimensions in velocity space. Since the rigid body moves in time, classical interpolation procedures near the rigid body become complicated and possibly inaccurate because of the arbitrary intersection of cells by the rigid body. We note that a cartesian cut cell method has been introduced in [11] to handle the moving object in the rarefied gas. A different technique has been used in [8], where the authors have used ghost point methods in a finite difference framework to treat moving boundaries. We refer also to the treatment of interfaces, for example, for multiphase flow problems in the framework of Lattice-Boltzmann schemes, see [18, 23] for a review and further references.

We use an immersed boundary type approach [21] to simulate the fluid-rigid body interactions. However, at variance with the original immersed boundary method, which dealt with an incompressible fluid, here we treat the interaction of a rarefied gas with a rigid body, see [2, 10] for immersed boundary approaches applied to kinetic equations. This means that we use a kinetic description of the gas, which is defined by a distribution function, and has therefore many more degrees of freedom than a (compressible or incompressible) fluid. The interaction with the boundary is based on mass conservation, and exchange of momentum and energy. About the energy exchange, we assume that the heat capacity of the solid is much larger than the one of the gas, so that the temperature of the solid object will be assumed constant in time. The approach is based on the combination between grid-based and mesh-free methods: the information about the distribution function is stored in an arbitrary fixed grid on a given domain. For this, the computational domain is discretized by a discrete set of fixed grid points which do not need to be regularly distributed. Moreover, the boundaries are also approximated by a discrete set of boundary points. On the boundary points we apply the boundary conditions. If boundaries move, the boundary particles also move with them.

In the present approach the rigid body overlaps the gas grid points. We do not consider those gas-grid-points which are overlapped by the rigid body in the computation and define them as in-active points. The non-overlapped points are defined as active points. All boundary points are defined as active points (refer to Figure 1). Therefore, the distribution of the active grid points is not uniform in the vicinity of a rigid boundary, even if we use a regular lattice for the gas grid points. Moreover,

it is varying over time as the rigid body moves. A moving least squares approach (later on referred as MLS) is a particularly suitable interpolation procedure in such a situation not requiring any special treatment. One only has to determine the overlapping and non-overlapping points and to update the kinetic distribution function from the active points with the help of the MLS-interpolation procedure. This process continues until the end of the simulation.

We finally note that in the present paper we restrict to a first order algorithm. Higher order methods are under construction.

The paper is organised as follows. In Section 2 we present the BGK model for the Boltzmann equation and the Newton-Euler equations for rigid body motions. In Section 3 we present the semi-Lagrangian scheme for the BGK model. Moreover, in the same section we present the moving least squares approximation, activating/deactivating grid points, boundary conditions and the coupling algorithm for the rigid body motion and rarefied gases. In Section 4 we present numerical results in one and two space dimensions. Finally, in Section 5 some conclusion and an outlook are presented.

2. The model

We consider the BGK-model for rarefied gas dynamics and the Newton-Euler equations for the motion of the rigid body inside the gas.

2.1. BGK model for rarefied gas dynamics

Consider first the BGK equation for the distribution function of gas molecules denoted by $f = f(t, \mathbf{x}, \mathbf{v})$, $t \geq 0$, $\mathbf{x} \in \Omega \subset \mathbb{R}^d$, ($d = 1, 2$) and $\mathbf{v} = (v_x, v_y, v_z) \in \mathbb{R}^3$. It is given by

$$\frac{\partial f}{\partial t} + \mathbf{v} \cdot \nabla f = \frac{1}{\tau}(M[f] - f) \quad (2.1)$$

with initial value $f(0, \mathbf{x}, \mathbf{v}) = f_0(\mathbf{x}, \mathbf{v})$ and boundary conditions discussed later. For the numerical examples we consider one- and two-dimensional spatial geometries and use suitable reduction procedures for the BGK equation, see [9, 14].

Here τ is the relaxation time and $M[f]$ is the local Maxwellian given by

$$M[f] = \frac{\rho}{(2\pi RT)^{3/2}} \exp\left(-\frac{|\mathbf{v} - \mathbf{U}|^2}{2RT}\right), \quad (2.2)$$

ρ, \mathbf{U}, T are macroscopic quantities: Density, mean velocity and temperature, R is the universal gas constant.

These macroscopic quantities are computed from $f(t, \mathbf{x}, \mathbf{v})$ in the following way. Let $\phi(\mathbf{v}) = \left(1, \mathbf{v}, \frac{|\mathbf{v}|^2}{2}\right)$ be the collision invariants. The moments are defined by

$$(\rho, \rho\mathbf{U}, E) = \int_{\mathbb{R}^3} \phi(\mathbf{v})f(t, \mathbf{x}, \mathbf{v})d\mathbf{v}. \quad (2.3)$$

E is the total energy density and it is related to the temperature through the internal energy

$$e(t, \mathbf{x}) = \frac{3}{2}RT, \quad \rho e = E - \frac{1}{2}\rho|\mathbf{U}|^2.$$

2.2. Newton-Euler equations for rigid body motion

The mechanical properties of the rigid body are uniquely defined by its mass and its moment of inertia, and its displacement is determined by the position of the center of mass and its orientation. The dynamics of the rigid body is determined by the Newton-Euler equations

$$M \frac{d\mathbf{V}}{dt} = \mathbf{F}, \quad [I] \cdot \frac{d\boldsymbol{\omega}}{dt} + \boldsymbol{\omega} \times ([I] \cdot \boldsymbol{\omega}) = \mathbf{T}, \quad (2.4)$$

where M is the total mass of the body S with center of mass \mathbf{X}_c , \mathbf{V} is the translational velocity of the center of mass, while $\boldsymbol{\omega}$ denotes the angular velocity vector describing the rotation around an axis passing through \mathbf{X}_c . \mathbf{F} is the translation force, \mathbf{T} is the torque and $[I]$ is the barycentric moment of inertia.

The center of mass of the rigid body are obtained by

$$\frac{d\mathbf{X}_c}{dt} = \mathbf{V}. \quad (2.5)$$

Finally, the velocity of the rigid body is given by $\mathbf{U}_w = \mathbf{V} + (\mathbf{x} - \mathbf{X}_c) \times \boldsymbol{\omega}$, $\mathbf{x} \in S$.

The force \mathbf{F} and torque \mathbf{T} are computed according to

$$\mathbf{F} = \int_{\Gamma_S} (-\boldsymbol{\varphi} \cdot \mathbf{n}) dA, \quad \mathbf{T} = \int_{\Gamma_S} (\mathbf{x} - \mathbf{X}_c) \times (-\boldsymbol{\varphi} \cdot \mathbf{n}) dA, \quad (2.6)$$

where \mathbf{n} is the unit boundary normal vector of the rigid body pointing towards the gas domain and $\boldsymbol{\varphi}$ is the pressure tensor given by

$$\boldsymbol{\varphi} = \int_{\mathbb{R}^3} (\mathbf{v} - \mathbf{U}_w) \otimes (\mathbf{v} - \mathbf{U}_w) f(t, \mathbf{x}, \mathbf{v}) d\mathbf{v}. \quad (2.7)$$

Γ_S denotes the boundary of S , see Figure 1.

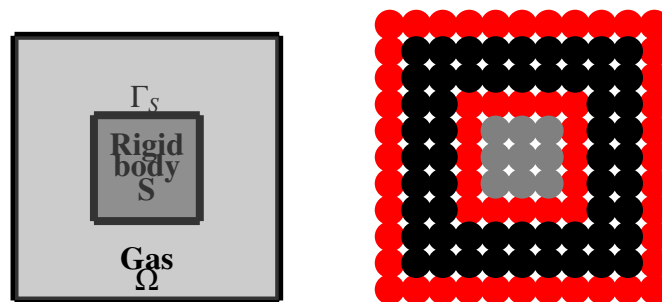


Figure 1. Rigid body S with boundary Γ_S immersed in the gas. Black and gray circles are active and non-active interior grid points, respectively and red circles are boundary grid points, which are always active.

In this paper we limit to study one and two dimensional problems. In 1D the rigid body does not rotate, so the moment of inertia does not play any role. In 2D the center of mass is determined by two coordinates, $\mathbf{X}_c = (X_c, Y_c)$, the only non zero component of the angular velocity vector is the out of

plane z -component ω , and the barycentric moment of inertia is a scalar which can be computed from the mass distribution of the object:

$$I = \int_S (\mathbf{x} - \mathbf{X}_c)^2 \rho(\mathbf{x}) d\mathbf{x}.$$

In all our tests we assume the mass is uniformly distributed in the body of the object (which is a surface in 2D), therefore the moment of inertia depends only on the mass and the geometry of the object.

3. Numerical methods

We describe a Semi-Lagrangian scheme with least squares interpolation for the BGK equation for three dimensional velocity space and two dimensional physical space.

3.1. Semi-Lagrangian scheme for BGK model

We consider a constant time step Δt , a uniform mesh in velocity space with mesh size Δv and a, in general, non-uniform mesh with average spacing Δx in physical space. The time discretization is denoted $t_n = n\Delta t, n = 0, 1, \dots$. The space discretization is obtained by filling (regular or irregular) grid points $\mathbf{x}_i = (x_i, y_i) \in \Omega \subset \mathbb{R}^2, i = 1, \dots, N_x$, where N_x is the total number of grid points in physical space. We note that the N_x grid points include interior as well boundary points. The interior grid points are fixed and located inside the whole computational domain including the moving object. In contrast, the boundary points are fixed to the boundaries, that means moving on the boundaries of the moving object and fixed on the fixed boundaries. The interior grid points are distinguished according to whether they are overlapping with the moving body or not. In the first case they are called *non-active* points, otherwise *active* points. See Figure 1 for an illustration. Moreover, we consider an even number N_v of velocity grid points in each direction and a uniform velocity grid size Δv in all directions. We assume the distribution function is negligible for $|v_{x,y,z}| > v_{max} = \frac{N_v \Delta v}{2}$. The uniform velocity grids are denoted by v_j, v_k and v_l in x, y and z directions, respectively, where $v_j = -v_{max} + (j - 1)\Delta v, j = 1, \dots, N_v + 1$. Similarly, we define v_k and v_l for $k, l = 1, N_v + 1$.

Let $f_{jkl} = f_{jkl}(t, \mathbf{x}) = f(t, \mathbf{x}, v_j, v_k, v_l)$ and $f_{ijkl} = f_{ijkl}(t) = f(t, \mathbf{x}_i, v_j, v_k, v_l)$. The evolution equation of $f_{jkl}(t, \mathbf{x})$ along the characteristics between time steps n and $n + 1$, i.e., for $t \in [t_n, t_{n+1}]$, is calculated from the Lagrangian form of the discrete-velocity BGK model

$$\frac{df_{jkl}}{dt} = \frac{1}{\tau} (M_{ijkl}[f] - f_{jkl}) \quad (3.1)$$

$$\frac{dx}{dt} = v_j, \quad (3.2)$$

$$\frac{dy}{dt} = v_k, \quad (3.3)$$

with final conditions

$$(x, y)(t_n) = (\tilde{x}, \tilde{y}), \quad f_{jkl}(t_n) = f_{ijkl}^n(\tilde{x}, \tilde{y}) = \tilde{f}_{jkl}^n \quad (3.4)$$

together with appropriate boundary conditions for f_{jkl} at boundary points.

Here $M_{ijkl}[f]$ is still the local Maxwellian having the moments of f_{jkl} .

We consider the implicit Euler scheme for the above equations, which reads

$$f_{ijkl}^{n+1} = \tilde{f}_{ijkl}^n + \frac{\Delta t}{\tau} (M_{ijkl}^{n+1}[f] - f_{ijkl}^{n+1}), \quad (3.5)$$

and

$$x_i^{n+1} = \tilde{x} + v_j \Delta t, \quad y_i^{n+1} = \tilde{y} + v_k \Delta t \quad (3.6)$$

for $j, k, l = 1 \dots, N_v + 1$ and all active interior points.

The semi-Lagrangian method now consists of three steps:

(i) First, we determine \tilde{x} and \tilde{y} from the backward characteristics $\tilde{x} = x_i^{n+1} - v_j \Delta t$, $\tilde{y} = y_i^{n+1} - v_k \Delta t$. Then reconstruct the function \tilde{f}_{ijkl}^n at (\tilde{x}, \tilde{y}) . At t^n all values f_{ijkl}^n are known for all active points and boundary points. At (\tilde{x}, \tilde{y}) we have to interpolate \tilde{f}_{ijkl}^n . One can use any interpolation formula. In this paper we use a least squares approximation for the reconstruction. This is presented in the next section.

(ii) In the second step we obtain M_{ijkl}^{n+1} . Since M_i^{n+1} and f_i^{n+1} give the same conservative moments, we multiply the above discrete equation by the collisional invariants $\phi(\mathbf{v})$ and sum over all velocities. We get

$$\rho_i^{n+1} = \sum_{j,k,l=1}^{N_v+1} \tilde{f}_{ijkl}^n \Delta v^3, \quad (\rho_i \mathbf{U}_i)^{n+1} = \sum_{j,k,l=1}^{N_v+1} \mathbf{v}_j \tilde{f}_{ijkl}^n \Delta v^3, \quad (3.7)$$

$$E_i^{n+1} = \frac{1}{2} \sum_{j,k,l=1}^{N_v+1} (v_j^2 + v_k^2 + v_l^2) \tilde{f}_{ijkl}^n \Delta v^3. \quad (3.8)$$

Once the moments are known, we can compute the Maxwellian at the new time.

(iii) Finally, we update the density function by

$$f_{ijkl}^{n+1} = \frac{\tau \tilde{f}_{ijkl}^n + \Delta t M_{ijkl}^{n+1}}{\tau + \Delta t}. \quad (3.9)$$

3.2. Time integration for the Newton-Euler equations

We solve the Newton-Euler equations by the explicit Euler method in time. The time step is the same as the time step of the BGK model. This means in particular, that the time step in the BGK model is chosen according to the stability requirements for the explicit Euler scheme for the Newton-Euler equations.

3.3. Moving least squares approximation

In this subsection we describe the least squares approximation of a function in a two-dimensional computational domain $\Omega \cup \Gamma \in \mathbb{R}^2$, where Γ is the boundary. As described above, we distinguish between the grid points on the boundary Γ and the interior grid points in Ω . The interior and boundary grid points are distinguished by assigning different flags, see Figure 1. Consider first the interior grid points (x_i, y_i) in Ω with average spacing Δx . They are chosen at the beginning of the calculation and are not moved. Those grid points overlapping with the moving body are *non-active*, the others are *active*.

Let $f(x, y)$ be a scalar function and f_i its values at (x_i, y_i) . We consider the problem of approximating the function $\tilde{f} = f(\tilde{x}, \tilde{y})$ at (\tilde{x}, \tilde{y}) from the values of its neighboring points. We associate a weight

function such that nearby particles have more and far away particles have less influence. Therefore, one can choose any distance function as a weight function which decays as the distance goes to infinity. In this paper we have considered a Gaussian function, but other choices are possible (see for example [26, 33] for other classes of weight functions). In order to limit the number of neighboring points we consider only the neighbors inside a circle of radius h with center (\tilde{x}, \tilde{y}) . We choose as radius h some factor of the average spacing Δx , such that we have at least a minimum number of neighbors for the least-squares approximation, even next to the boundary. In case of regular grid and far from the boundary, one might consider using smaller values of h . Such adaptive choice of h has been considered, for example, in [17]. For the sake of simplicity, we have chosen a constant $h = 3.1 \Delta x$ in this paper, which gives a sufficiently large number of neighbours even near concave boundaries (as is the case of the Example 6 in the last section). The use of adaptive values of h is left to future investigation. We sort the neighboring points from 1 to m with respect to distance, such that the neighbour index 1 is the nearest neighbor of (\tilde{x}, \tilde{y}) . With a slight abuse of notation, let $P(\tilde{x}, \tilde{y}; h) = \{(x_j, y_j), j = 1, \dots, m(h)\}$ denote the set of neighbor points of (\tilde{x}, \tilde{y}) inside the disc of radius h . We note that the number m of neighbours depends on (\tilde{x}, \tilde{y}) and h . In all calculations we have considered the following truncated Gaussian weight function

$$w(x_i - \tilde{x}, y_i - \tilde{y}; h) = \begin{cases} \exp(-\alpha \frac{(x_i - \tilde{x})^2 + (y_i - \tilde{y})^2}{h^2}), & \text{if } \frac{\sqrt{(x_i - \tilde{x})^2 + (y_i - \tilde{y})^2}}{h} \leq 1 \\ 0, & \text{else,} \end{cases}$$

with α a user defined positive constant, chosen here as $\alpha = 6$, so that the the influence of far neighbor grid points is negligible. This choice has been suggested from previous experience [17, 28]. It would be interesting to investigate what is the optimal choice of the parameters, or even to adopt a different class of weight functions. This is left to future investigation.

In order to approximate the function we consider the m Taylor's expansions of $f(x_j, y_j)$ around (\tilde{x}, \tilde{y})

$$f(x_j, y_j) = f(\tilde{x}, \tilde{y}) + \frac{\partial}{\partial x} f(\tilde{x}, \tilde{y})(x_j - \tilde{x}) + \frac{\partial}{\partial y} f(\tilde{x}, \tilde{y})(y_j - \tilde{y}) + e_j, \quad (3.10)$$

for $j = 1, \dots, m$, where e_j is the error in the Taylor's expansion. We first assume that \tilde{f} approximates the nearest point f_1 . In other words $e_1 = 0$. The unknowns $\tilde{f}, \frac{\partial \tilde{f}}{\partial x}, \frac{\partial \tilde{f}}{\partial y}$ are computed by minimizing the error e_j for $j = 2, \dots, m$ and setting the constraint $e_1 = 0$. To solve this constraint least-squares problem, we use the constraint to rewrite the equations in the form

$$\begin{aligned} f_2 - f_1 &= \frac{\partial \tilde{f}}{\partial x}(x_2 - x_1) + \frac{\partial \tilde{f}}{\partial y}(y_2 - y_1) + e_2 \\ &\vdots = \vdots \\ f_m - f_1 &= \frac{\partial \tilde{f}}{\partial x}(x_m - x_1) + \frac{\partial \tilde{f}}{\partial y}(y_m - y_1) + e_m \end{aligned} \quad (3.11)$$

The system of equations can be written in matrix form as

$$\mathbf{e} = \mathbf{b} - \mathbf{M}\mathbf{a}, \quad (3.12)$$

where $\mathbf{e} = [e_2, \dots, e_m]^T$, $\mathbf{a} = [\frac{\partial \tilde{f}}{\partial x}, \frac{\partial \tilde{f}}{\partial y}]^T$, $\mathbf{b} = [f_2 - f_1, \dots, f_m - f_1]^T$ and

$$M = \begin{pmatrix} x_2 - x_1 & y_2 - y_1 \\ \vdots & \vdots \\ x_m - x_1 & y_m - y_1 \end{pmatrix}.$$

For $m > 3$, this system of equations is over-determined for two unknowns $[\frac{\partial \tilde{f}}{\partial x}, \frac{\partial \tilde{f}}{\partial y}]^T$. The unknowns \mathbf{a} are obtained from the weighted least squares method by minimizing the quadratic form

$$J = \sum_{j=2}^m w_j e_j^2 = (\mathbf{M}\mathbf{a} - \mathbf{b})^T \mathbf{W}(\mathbf{M}\mathbf{a} - \mathbf{b}), \quad (3.13)$$

where $\mathbf{W} = w_j \delta_{jk}$, $k = 2, \dots, m$ is the diagonal matrix. The minimization of J yields

$$\mathbf{a} = (\mathbf{M}^T \mathbf{W} \mathbf{M})^{-1} (\mathbf{M}^T \mathbf{W}) \mathbf{b}. \quad (3.14)$$

Now from Eq. (3.10) with $e_1 = 0$ for the closest point x_1 we can compute the value of $f(\tilde{x}, \tilde{y})$ at (\tilde{x}, \tilde{y}) as

$$f(\tilde{x}, \tilde{y}) = f(x_1, y_1) - \frac{\partial \tilde{f}}{\partial x} (x_1 - \tilde{x}) - \frac{\partial \tilde{f}}{\partial y} (y_1 - \tilde{y}) \quad (3.15)$$

since $\frac{\partial \tilde{f}}{\partial x}$ and $\frac{\partial \tilde{f}}{\partial y}$ are now known. We note that higher order approximations are obtained by using higher order Taylor's expansion in (3.10). We refer to [28] for details.

In the above least-squares approximation a function is approximated at an arbitrary point from its neighboring points and the distribution of these points can be arbitrary. Such a straightforward least-squares approximation leads to a central difference scheme. In case of discontinuities in the solution, this will lead to numerical oscillations and one has to introduce additional numerical viscosity. This can be done in the least squares framework by adopting a suitably modified version of that approach using an upwind reconstruction.

Moreover, we note that for the stabilization of higher order approximations a WENO-type reconstruction can be used, see e.g., [1, 32, 33], where WENO approximations with least squares approaches have been developed for regular and irregular grids.

3.4. Activating/deactivating grid points

For the simulation of the interaction of the rigid body motion with the gas, we overlap the region defined by the rigid body and the region where the BGK model is computed. Those grid points in the gas phase which are overlapped by the rigid body during the motion are assigned as non-active grids and the others as active grids. The non-active grids are taken out of the numerical process and sorted out from the neighboring lists in the least-squares approximations. After the rigid body movement, some of the active grids will be overlapped by the rigid body and then redefined as non-active grids. In turn, some of non-active grids will be out of the overlapping zone of the rigid body and will be reactivated again for the numerical process. During this process we need to update the distribution function $f(t, \mathbf{x}, \mathbf{v})$ on the newly activated grid. This can be obtained from its neighboring active grid points using the least squares method from above.

3.5. Boundary conditions

On the solid boundary as well as on the moving rigid object boundaries we apply diffuse reflection boundary conditions. The boundary particles are sitting on the boundaries and all boundary points having contact with the gas phase are defined as *active* points. The boundary particles move with the boundary velocities. The boundary conditions are applied on the boundaries of the computational domain as well as on the surface of rigid body. Let $\rho_w, T_w, \mathbf{U}_w$ and \mathbf{n} be the density, temperature, velocity and \mathbf{n} unit normal of the wall and the surface of the rigid body. The wall normal vector \mathbf{n} points towards the gas domain.

For $(\mathbf{v} - \mathbf{U}_w) \cdot \mathbf{n} < 0$ we obtain the distribution function on the wall f_w^{n+1} from the evolution equation. For $(\mathbf{v} - \mathbf{U}_w) \cdot \mathbf{n} > 0$ the distribution function is the Maxwellian with parameters ρ_w, T_w and \mathbf{U}_w , which is given by

$$M_w^{n+1} = \frac{\rho_w}{(2\pi RT_w)^{3/2}} \exp\left(-\frac{|\mathbf{v} - \mathbf{U}_w|^2}{2RT_w}\right). \quad (3.16)$$

We note that the density ρ_w is not known and is determined by assuming the net flux across the wall or surface is zero. This means, we have

$$\int_{\mathbb{R}^3, (\mathbf{v} - \mathbf{U}_w) \cdot \mathbf{n} > 0} (\mathbf{v} - \mathbf{U}_w) \cdot \mathbf{n} M_w^{n+1} d\mathbf{v} + \int_{\mathbb{R}^3, (\mathbf{v} - \mathbf{U}_w) \cdot \mathbf{n} < 0} (\mathbf{v} - \mathbf{U}_w) \cdot \mathbf{n} f_w^{n+1} d\mathbf{v} = 0. \quad (3.17)$$

Hence, from (3.16) and (3.17) we obtain

$$\rho_w = -\frac{\int_{\mathbb{R}^3, (\mathbf{v} - \mathbf{U}_w) \cdot \mathbf{n} < 0} (\mathbf{v} - \mathbf{U}_w) \cdot \mathbf{n} f_w^{n+1} d\mathbf{v}}{\int_{\mathbb{R}^3, (\mathbf{v} - \mathbf{U}_w) \cdot \mathbf{n} > 0} (\mathbf{v} - \mathbf{U}_w) \cdot \mathbf{n} \frac{1}{(2\pi RT_w)^{3/2}} \exp\left(-\frac{|\mathbf{v} - \mathbf{U}_w|^2}{2RT_w}\right) d\mathbf{v}}. \quad (3.18)$$

3.6. Coupling rigid body motion and the rarefied gas

After calculation of the new density function f_j^{n+1} we first compute the pressure tensor (2.7) on all boundary points of the rigid body. Then we approximate the force and torque on the rigid body according to (2.6). We obtain the translational and rotational velocities and then move the boundary points and the center of mass accordingly. Finally, we update the normal vector \mathbf{n} . The new velocity \mathbf{U}_w is used to apply the boundary conditions for solving the BGK model. In summary, we use the following coupling algorithm:

- (i) Generate initial grid points with flags as interior and boundary grids and prescribe the initial conditions in the gas as well as in the solid phases.
- (ii) Determine the active and non-active grids in the gas phase.
- (iii) Update newly activated grid points in the gas phase with the help of interpolations from its active neighbors.
- (iv) Solve the BGK model equation in the active grid points and apply boundary conditions on all boundary points.
- (v) Compute the force and torque on the boundary points of the rigid body.
- (vi) Solve the Euler-Newton equations and then get new positions, velocity and the unit normal of the boundary points of the rigid body.
- (vii) Goto (ii) and repeat until the final time is reached.

4. Numerical results

In the following we consider numerical examples in one and two space dimensions and three velocity dimensions. The test cases are given in dimensionless form but can be interpreted in SI-units.

4.1. Example 1: 1D moving piston, one way coupling

This problem has been considered in [10, 22] in a larger domain. We consider the one-dimensional spatial domain $\Omega = [0, 3 \times 10^{-3}]$. Initially the piston is positioned at $x = 1.5 \times 10^{-4}$. We considered the total number $N_x = 300$ grid points in physical space and in $N_v = 30$ grid points in every direction of velocity space. The left boundary moves with velocity

$$u_p = 10 * \sin\left(\frac{t}{10^{-6}}\right).$$

This is a one way coupling, since the motion of the piston is prescribed. We note that initially, some grid points less than $x = 1.5 \times 10^{-4}$ are overlapped by the piston. They are non-active points and the piston position and the right boundary points are the active grid points, see Figure 2 for physical setup of the problem.

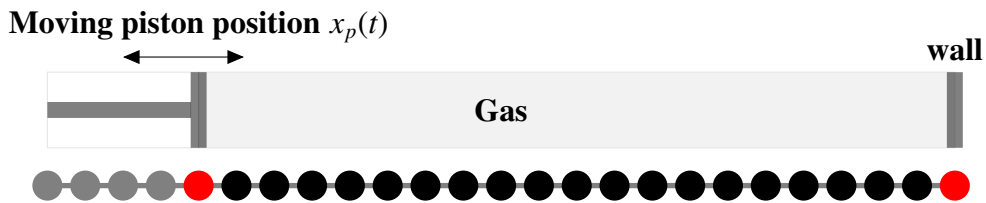


Figure 2. Geometrical set up for moving piston problem. The black circles are active grid points, grey circles are non-active grid points and red circles are boundary points.

When the piston starts to move in time the process of activating and deactivating of grid points continues throughout the simulation. We have considered the final time $t_{final} = 4 \times 10^{-6}$. The time step is $\Delta t = 10^{-9}$. The minimum and maximum limit of the velocity are $v_{min} = -1200$ and $v_{max} = 1200$. We have considered the Argon gas with diameter $d = 0.368 \times 10^{-9}$, Boltzmann constant $k_b = 1.3806 \times 10^{-23}$ and the universal gas constant $R = 208$. The initial temperature $T_0 = 270$, initial density $\rho_0 = 0.00018$ and the initial mean velocity $\mathbf{U}_0 = \mathbf{0}$. The corresponding Knudsen number is $Kn = \lambda/L = 0.215$, based on the characteristic length $L = 3 \times 10^{-3} - 1.5 \times 10^{-4}$, where λ is the mean free path defined by

$$\lambda = \frac{k_b}{\sqrt{2\pi\rho_0 R d^2}}. \quad (4.1)$$

To validate the numerical results of the semi-Lagrangian scheme for the BGK model, we compare it with the results of a numerical solution of the full Boltzmann equation via the DSMC method [3, 5, 19]. For a proper comparison of the BGK model and the DSMC code for the Boltzmann equation we have to relate the relaxation time τ and the mean free path, see [7], as

$$\tau = \frac{4\lambda}{\pi\bar{C}}, \quad (4.2)$$

where $\bar{C} = \sqrt{\frac{8RT_0}{\pi}}$. The corresponding relaxation time $\tau = 2.0634 \times 10^{-6}$. Nevertheless, by DSMC we solve the full Boltzmann equation, therefore differences in results may be due partially to the different models, and partially to the different numerical techniques adopted in the two cases.

Initially, the piston and the gas are at rest. The piston starts oscillating in time and disturbs the gas phase. A wave is formed which eventually creates a shock. The flow is a low Mach number flow and the DSMC results show strong fluctuations. Since the flow is unsteady one cannot take time averages of the DSMC simulations. Therefore, one has to perform several independent runs. In the DSMC simulations we have considered the same number of cells as in the BGK model. To reduce the statistical noise, we have considered 10.000 gas molecules per cells initially. Moreover, we have performed 500 independent runs. In Figures 3–6 we have plotted the density, temperature and velocity of the gas determined from both numerical methods. We can observe that the BGK and the DSMC solutions have very good agreements at all times. We note, that the statistical noise for the DSMC simulations is still observed even after 500 independent runs.

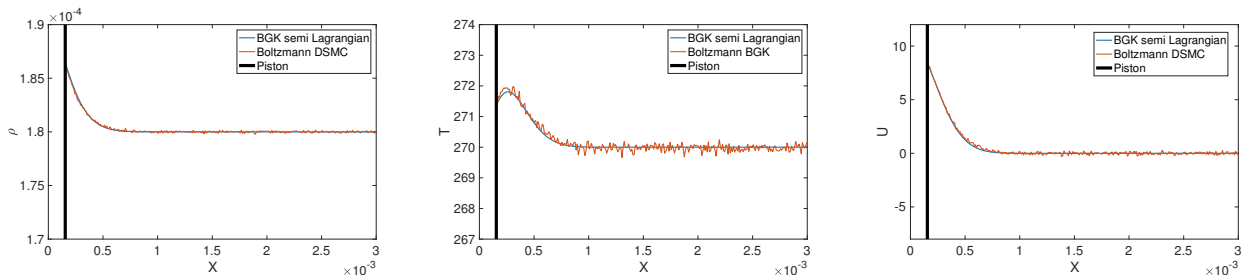


Figure 3. Comparison of BGK and DSMC at time $t = 1 \times 10^{-6}$.

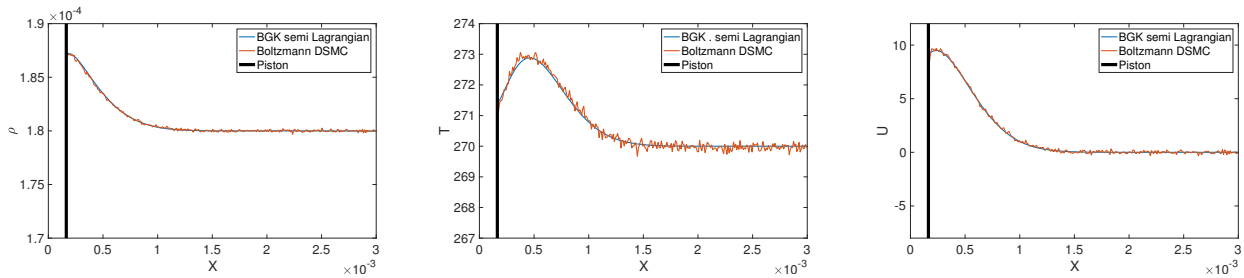


Figure 4. Comparison of BGK and DSMC at time $t = 2 \times 10^{-6}$.

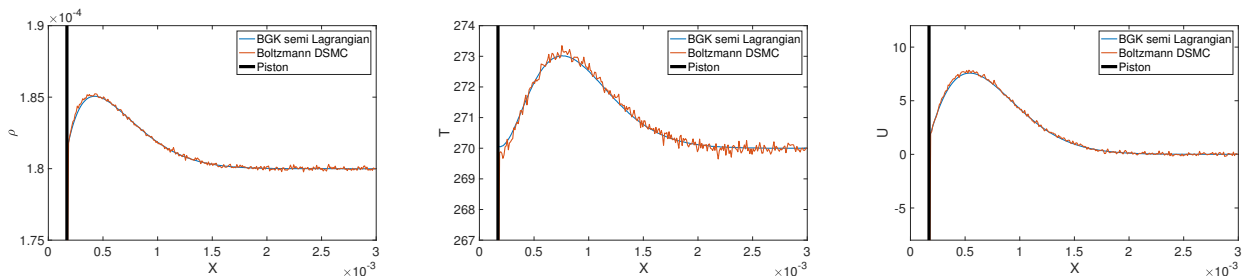


Figure 5. Comparison of BGK and DSMC at time $t = 3 \times 10^{-6}$.

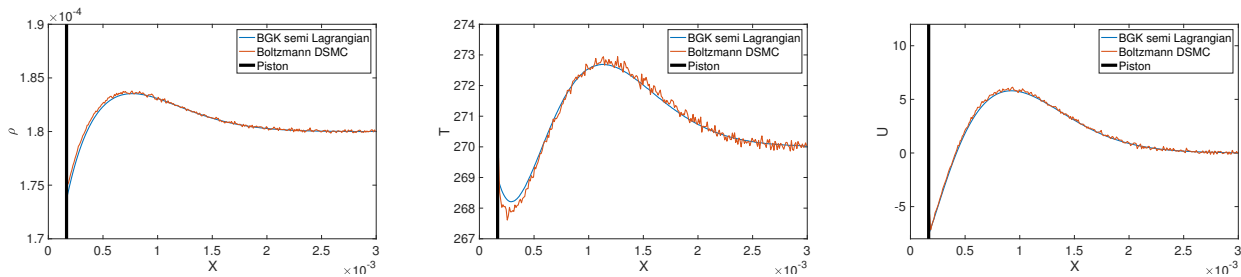


Figure 6. Comparison of BGK and DSMC at time $t = 4 \times 10^{-6}$.

In both methods we apply diffuse reflection boundary conditions on the piston with moving frame of reference $\mathbf{U}_w = (u_p, 0, 0)$, where u_p is the velocity of the piston, and the wall temperature $T_w = T_0$. Similarly, we apply diffuse reflection boundary conditions with zero wall velocity and a wall temperature equal to the initial temperature on the right boundary.

4.2. Example 2: 1D moving plate, two way coupling

In Example 1, the gas flow was influenced by the motion of the piston, but there was no any influence of the gas flow on the motion of the piston. In this example, we consider a two way coupling of both phases. The force exerted on the rigid body from the surrounding gas influences the motion of the rigid body and vice versa. We again consider a one dimensional physical space and three dimensional velocity space. We consider the physical domain $[-(L + l), (L + l)]$ as described in Figure 7 with $L = 1$ and $l = 0.1$, where $2l$ is the thickness of a plate which is driven by the pressure difference at its edges.

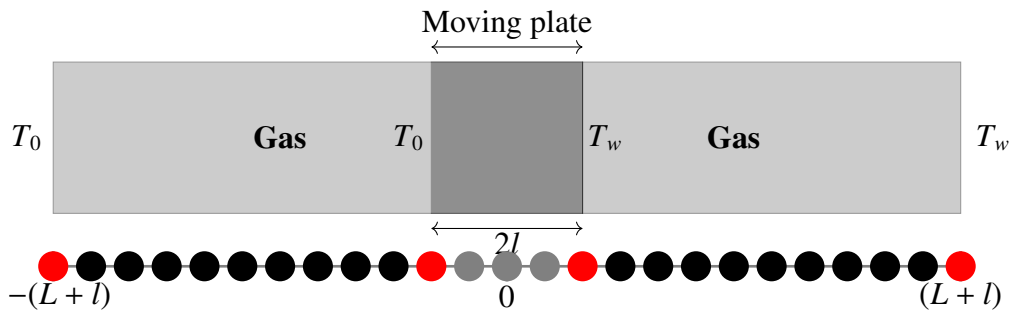


Figure 7. Schematic view of a plate separating to subdomains with different temperature. Like in the piston problem, the black circles are active grids, the grey circles are non-active grids and the red circles are boundary points.

Initially the plate is located at $(-0.1, 0.1)$ with center of mass $X_c = 0$, where the gas and the plate are at rest. This problem has been studied in [10]. We reconsider it as a benchmark problem since an analytical solution is available for the equilibrium state. We again consider a monatomic gas with parameters given in Example 1. The initial temperature is $T_0 = 270$ and the initial pressures P_0 are the same on both sides of the plate and are equal to 0.0386. The initial density ρ_0 is obtained from the equation of state. The initial Knudsen number is 0.08 based on the characteristic length $2L$ and the relaxation time $\tau = 5.398 \times 10^{-4}$. Moreover, we have considered different density ratios of the gas and the plate. The other parameters are the same as in the Example 1. We prescribe a higher temperature

$T_w = 330$ on the right side of plate and on the right boundary of the computational domain. On the left boundary of the plate and on the left boundary of the computational domain the temperature is kept to T_0 . Due to the high temperature on the right walls, the pressure on the right hand side starts to increase and the plate starts to move to the left hand side. The motion of the plate is computed from the Euler-Newton equations, where only a translational force is computed for the one dimensional case. Since the plate has two opposite normals ± 1 , from Eq. (2.6) the total force is given as the difference of pressure

$$F = (\varphi_{\text{left}} - \varphi_{\text{right}})A, \quad (4.3)$$

where A is the area of plate. The plate starts oscillating and finally reaches the equilibrium position [10]

$$x_{\text{equi}} = L \frac{(T_0 - T_w)}{(T_0 + T_w)} = -0.1. \quad (4.4)$$

The domain is discretized with $N_x = 300$ cells. The velocity grid is given by $N_v = 20$ cells for the BGK equations. The final time is 0.5. The time step is $\Delta t = 4 \times 10^{-6}$. The other parameters are the same as in Example 1. The explicit Euler method with the same time step as the time step for the BGK model is used for time integration of the Newton-Euler equations.

In this test case we have simulated a wide range of density ratios of gas and plate ranging from 1 to 10 up to 1 to 1000. In Figures 8 we have plotted the velocity of the plate with respect to time together with the exact equilibrium solution. One can observe, as expected, that a lower density plate reaches the equilibrium position earlier than the heavier plates. We remark, however, that the change of density ratio has been used just to change the mass of the plate. Indeed the dynamics depend only on the mass of the plate, not on its density. We choose to change the density just to allow a finite size of the plate (which is left unchanged in our simulations).

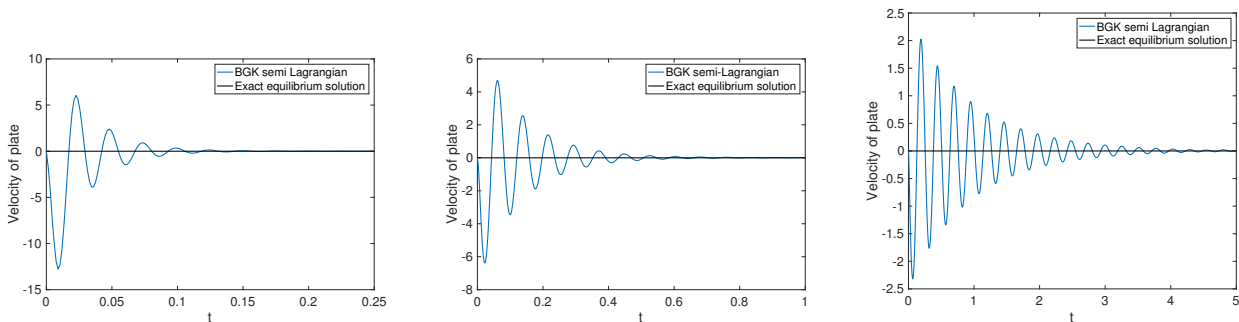


Figure 8. Comparison of velocity vs time of plate for density ratios of gas and plate 1 to 10 (top left) and 1 to 100 (top right) and 1 to 1000 (bottom) for $N_x = 500$ and $N_v = 20$.

Additionally, we have performed a convergence study for the case of a density ratio 1 to 50. The results for the plate position and velocity are reported in Figure 9. We note that for $N_x = 300$ we obtain an accurate approximation of the equilibrium value for the velocity, whereas, the equilibrium position still deviates from the analytical value. This is due to the first order error of the numerical scheme and an accumulation of very small numerical errors in velocity during the integration process. Increasing the number of grid points N_x in physical space, we obtain convergence towards the analytical

equilibrium position. We remark that a velocity grid with $N_v = 30$ gives almost the same results, see [27] for a numerical comparison.

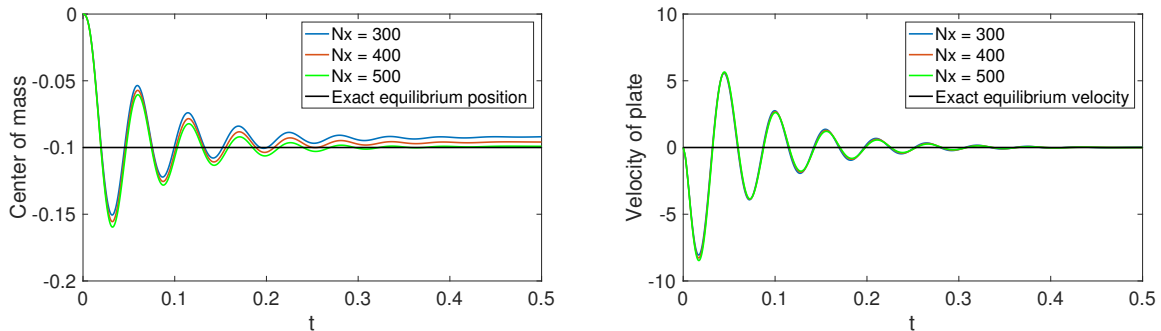


Figure 9. Comparison of position and velocity vs time of plate with the density ratio 1 to 50 for number of cells $N_x = 300, 400$ and 500 and $N_v = 20$.

We have further compared the solutions of the BGK model with the DSMC simulations for $N_x = 500$. In the case of the DSMC simulations we have again considered 400 gas molecules per cell initially. The boundary conditions and other parameters are the same in both methods. We have performed 50 independent runs. In Figure 10 we have plotted the position of the center of mass and the velocity of piston vs time. We note that the time evolution of the the DSMC solutions and the solutions obtained from the BGK semi Lagrangian method are very close to each other.

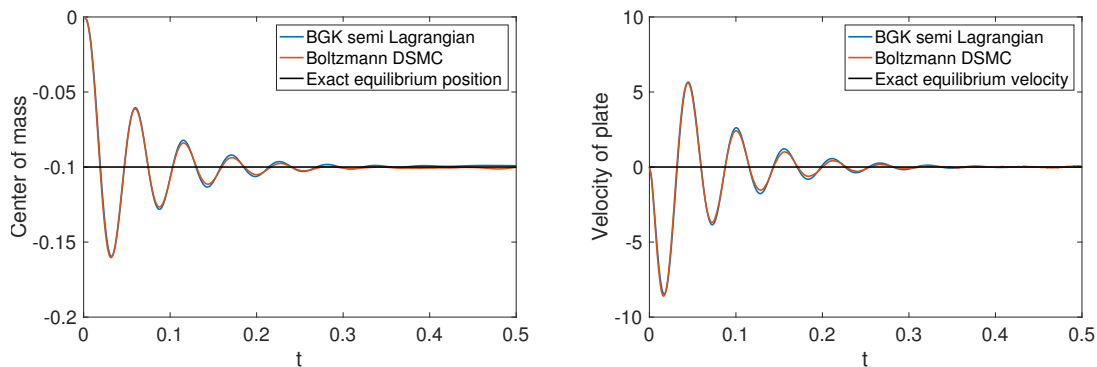


Figure 10. The solution for a Knudsen number of 0.08. Left: early-stage position of the center of mass of piston vs time. Right: velocity of the plate vs time with the density ratio 1 to 50. The red solid line represents the exact equilibrium position and the blue line represents the numerical values.

Furthermore, in Figure 11 we have plotted the temperature obtained from both methods for time $t = 0.1, 0.2$ and 0.4 . We see that at time $t = 0.1$ the temperature is not yet reaching equilibrium state, but after $t = 0.2$ the temperature of the gas on the left reaches the left wall temperature and the temperature of the gas on the right reaches the right wall temperature.

Similarly, we have plotted the velocity field of the gas on both sides of the plate in Figure 12 at different times for both methods. Here, we also observe, that the DSMC solutions fluctuate around the

BGK solutions.

Finally, in Figure 13, we have plotted the pressure obtained from the BGK model and the DSMC simulations. In the beginning, the pressure on the right increases due to the increase of the temperature. When it reaches $t = 0.1$ the pressure on the left is slightly larger than on the right side. At time $t = 0.2$ the pressure on the left is still larger, which is clearly visible in the figure. It fluctuates and finally reaches the equilibrium state, where it is equal on both sides. The relative error of the computation versus the analytical solution [10] is approximately 0.7% for pressure and 0.6% for density in the stationary state for the finest discretization $N_x = 500$.

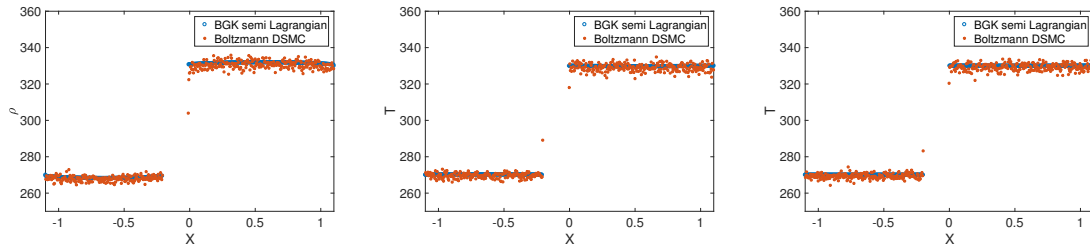


Figure 11. Temperature plots at times 0.1, 0.2 and 0.4 for the density ratio 1 to 50.

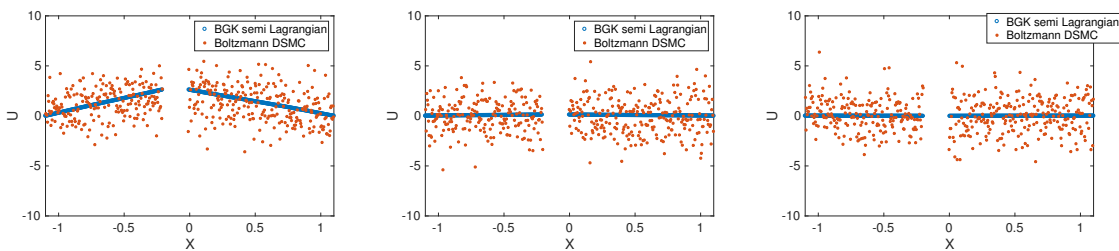


Figure 12. Velocity plots at times 0.1, 0.2 and 0.4 for the density ratio 1 to 50.

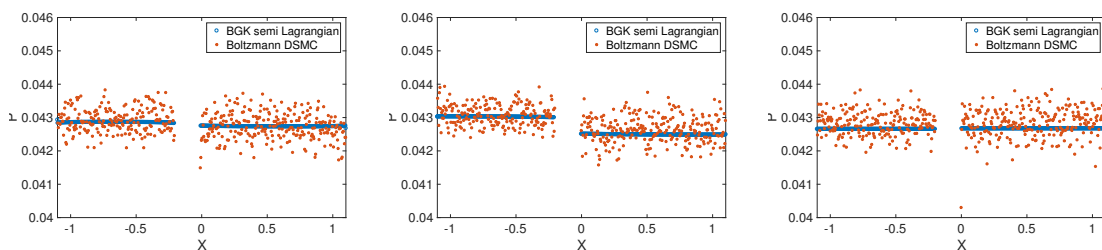


Figure 13. Pressure plots at times 0.1, 0.2 and 0.4 for the density ratio 1 to 50.

4.3. Example 3: 2D driven cavity problem

Here, we consider a spatially 2-dimensional problem with three-dimensional velocity space. The flow in a cavity driven by the velocity on the top is a widely used benchmark problem for testing and comparing numerical methods. We consider a micron size square cavity. The top wall has velocity

$$U_x = u_w, U_y = 0, \tag{4.5}$$

and on the other three walls we have $U_x = U_y = 0$. The temperature is kept constant at $T_0 = 270$ on all walls, the initial density is $\rho_0 = 1$, the wall velocity $u_w = 1$ and the gas constant is $R = 208$. The gas is again monatomic with parameters given as in the Example 1. The Knudsen number $Kn = 0.1$ is based on the characteristic length given by the size of the wall. Diffuse reflection boundary conditions are applied on all walls. In Figure 14 we have plotted the regular and irregular gridpoints used for the simulation. Figure 15 shows the velocity fields and the vorticity obtained from the BGK equation for regular as well as irregular grid points. We use 50×50 gridpoints and approximately the same number of irregular grid points is generated. The time step is chosen as 5×10^{-11} . The simulation was stopped after time $t = 4 \times 10^{-7}$. Moreover, Figure 16 compares the x - and y -velocity components along the center lines in y - and x -direction, respectively. We observed that the solutions obtained from the regular and irregular grids are almost the same.

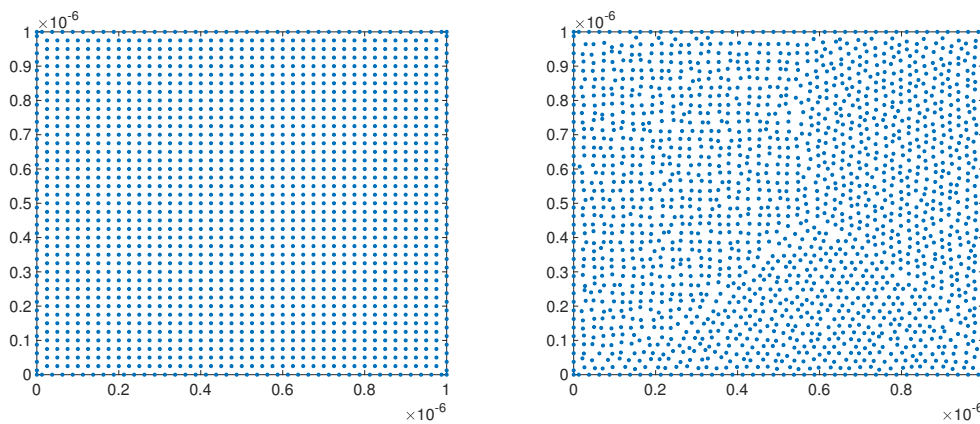


Figure 14. Regular grids (left) and irregular grids (right) for solving BGK model.

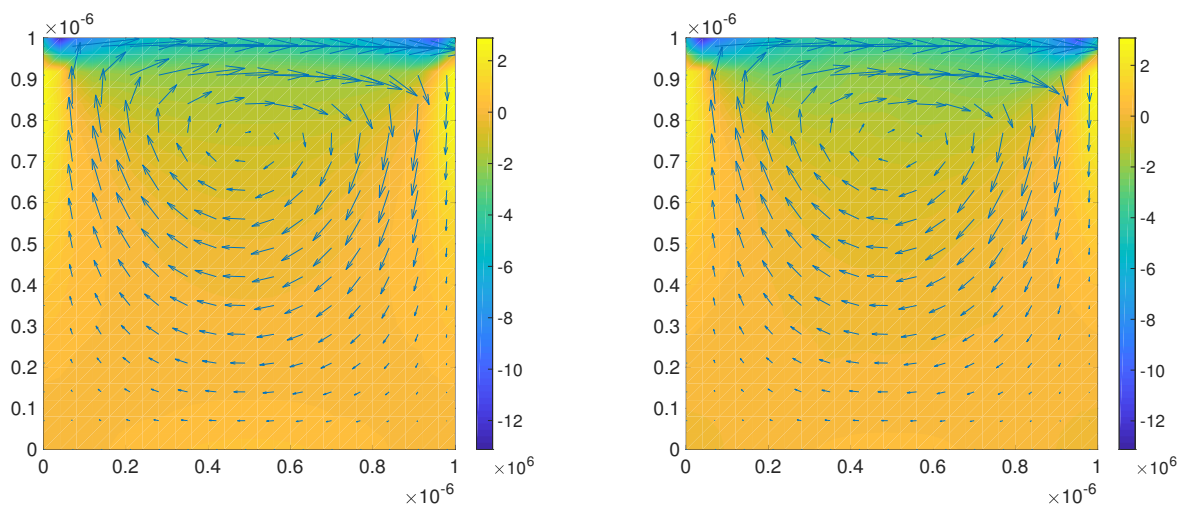


Figure 15. Velocity field and out-of-plane vorticity obtained from regular(left) and irregular (right) grids from the BGK model.

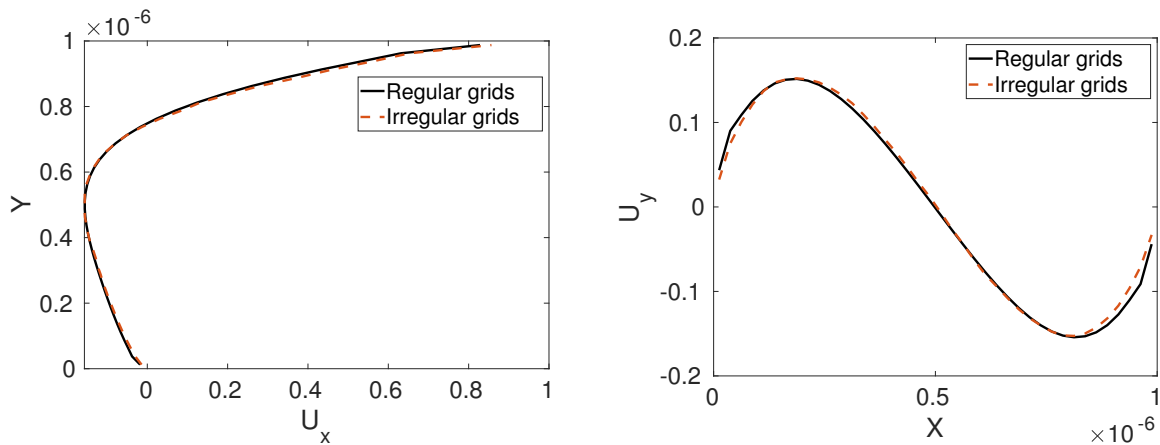


Figure 16. Comparison of the velocity components obtained in regular as well irregular grids in the center of cavity. Left: x -component of velocity along the center line in y -direction. Right: y -component of velocity along center line in x -direction for $u_w = 1$.

Moreover, we compare the semi-Lagrangian scheme for the BGK model with DSMC simulations for the Boltzmann equation for this example. The mean free path and the relaxation time are chosen according to Eq. (4.2). First, we choose the velocity of the upper wall as $u_w = 1$ in positive x -direction. In the DSMC simulations we have taken the same time step and the same number of cells as in the BGK model. For the DSMC simulations we have also applied diffuse reflection boundary conditions on all walls. The time steps and the gas parameters are the same as in the BGK model. In this case we look at the steady state solution. Therefore, unlike in the earlier two examples, we do not perform independent runs, but time averages.

In Figure 17 on the left we have run the DSMC simulation up to 10^5 time steps, where the last 9×10^4 time steps are used for the averaging over the samples. In this case the fluctuations dominate the flow field. In Figure 17 on the right 3×10^6 time steps are used for the averaging over the samples.

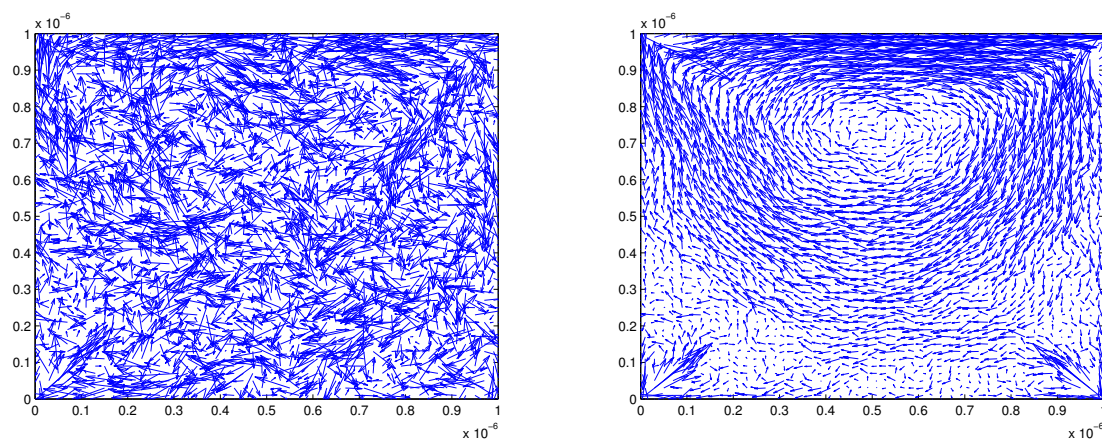


Figure 17. DSMC simulations with 9×10^4 samples (left) and with 3×10^6 samples (right) for $u_w = 1$.

In Figure 18 we have plotted the x -velocity component U_x along the central vertical line as well as the y -velocity component U_y along the central horizontal line for the case with 9×10^4 samples. We observe again the highly oscillating DSMC results compared to the BGK solutions.

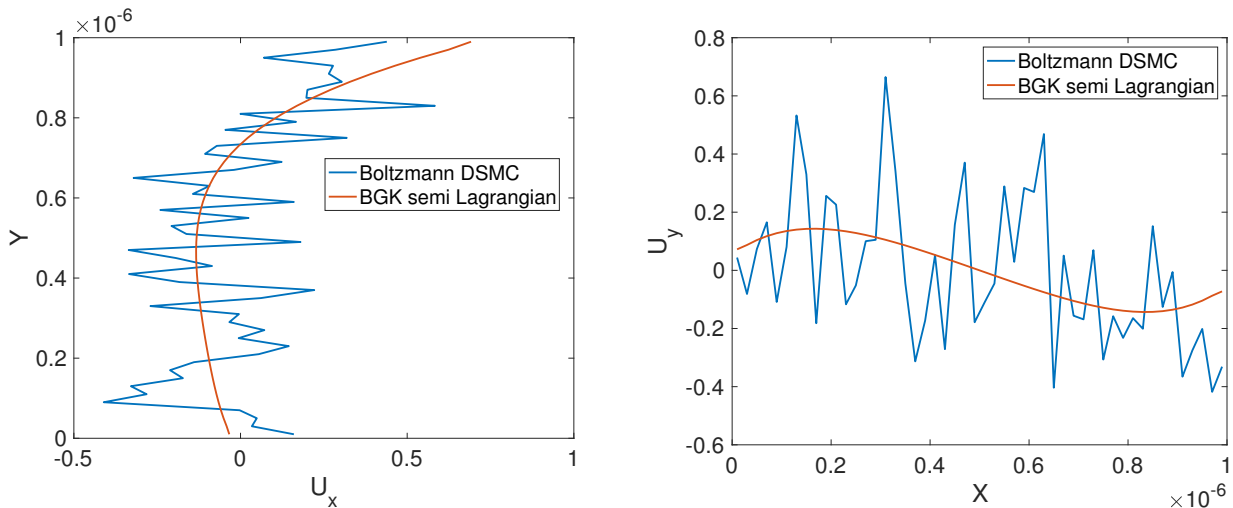


Figure 18. Left: x -component of velocity along the center line in y -direction and Right: y -component of velocity along center line in x -direction with 9×10^4 sampling for $u_w = 1$.

Similarly, in Figure 19 we have plotted the x -velocity component U_x along the central vertical line as well as y -velocity component U_y along the central horizontal line for 3×10^6 samples. The DSMC results are improving, but still fluctuating around the BGK solutions.

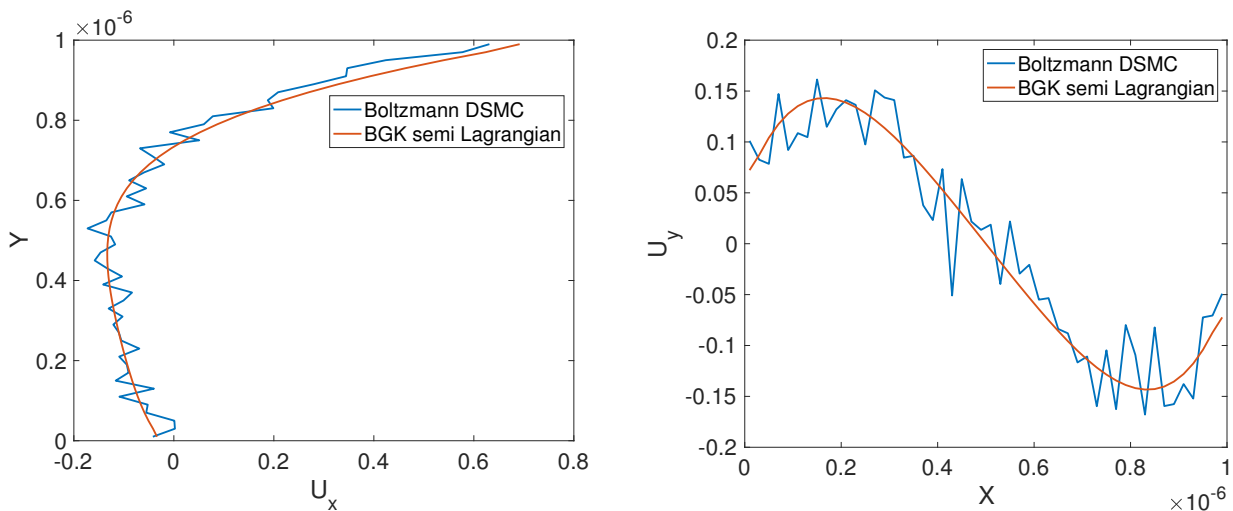


Figure 19. Left: x -component of velocity along the center line in y -direction and Right: y -component of velocity along center line in x -direction with 3×10^6 sampling for $u_w = 1$.

4.4. Example 4: Moving a 2d rigid body in rarefied gas

This example is the direct extension of Example 1 into two space dimensions. Here we have used a Chu reduction [9, 14] to reduce the dimension of the velocity space from three to two. We have taken this problem from the paper by Frangi et al. [13], where the authors have studied the biaxial accelerometer produced by STMicroelectronics with a surface micro-machining process. The authors have analysed the problem by considering a two-dimensional simplification. In Figure 20 we have sketched the computational domain in details. The shuttle lies initially in the middle of the domain. In the rest of the domain a gas flow is taking place. The shuttle oscillates with the velocity $\cos(2\pi f_0 t)$, where f_0 is the frequency. The parameters mentioned in the Figure 20 are $L_1 = 19.2 \times 10^{-6}$, $d_1 = 4.2 \times 10^{-6}$, $d_2 = 2.6 \times 10^{-6}$, $d_3 = 5 \times 10^{-6}$, $d_4 = 3.9 \times 10^{-6}$, $d_5 = 18.8 \times 10^{-6}$. In [13] the frequency has been taken $f_0 = 4400$ Hz, but with this frequency, the shuttle crosses the upper and lower boundaries. Therefore, we have chosen $f_0 = 40 \times 4400$ Hz such that the maximum amplitude of the shuttle is half of the distance d_2 . The initial pressure of the gas is equal to 0.1 bar, which corresponds to initial density $\rho_0 = 0.1641$. The initial distribution f_0 of the gas is the Maxwellian with zero mean velocity, initial temperature $T_0 = 293$ and initial density ρ_0 . The diffuse reflection boundary condition with wall temperature T_0 is applied on the solid lines and a far field boundary condition f_0 is applied on the dotted lines. We note that here, we solve the real motion of the shuttle, while in [13] the authors solve the stationary equations with assigned non zero velocity on the boundary.

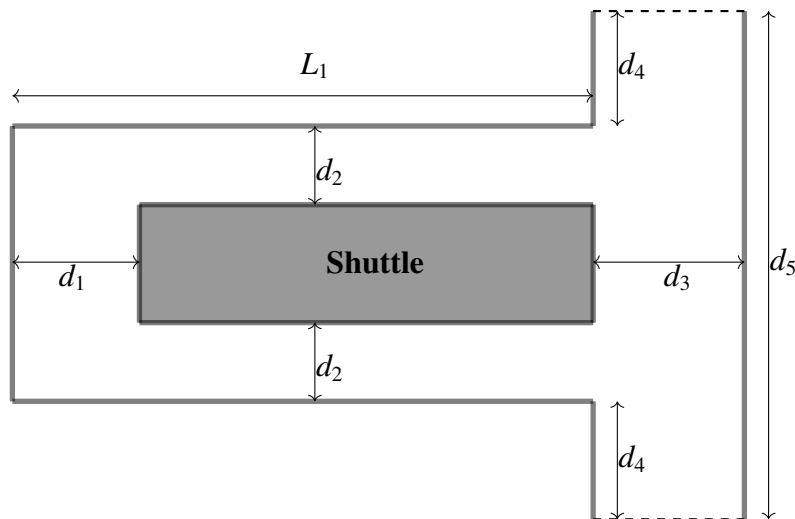


Figure 20. Geometry setup for moving 2D shuttle.

In Figure 21 we have plotted the velocity vector fields as well as x - and y - components of the velocity at times $t = 1.2 \times 10^{-6}$ and $t = 3.6 \times 10^{-6}$.

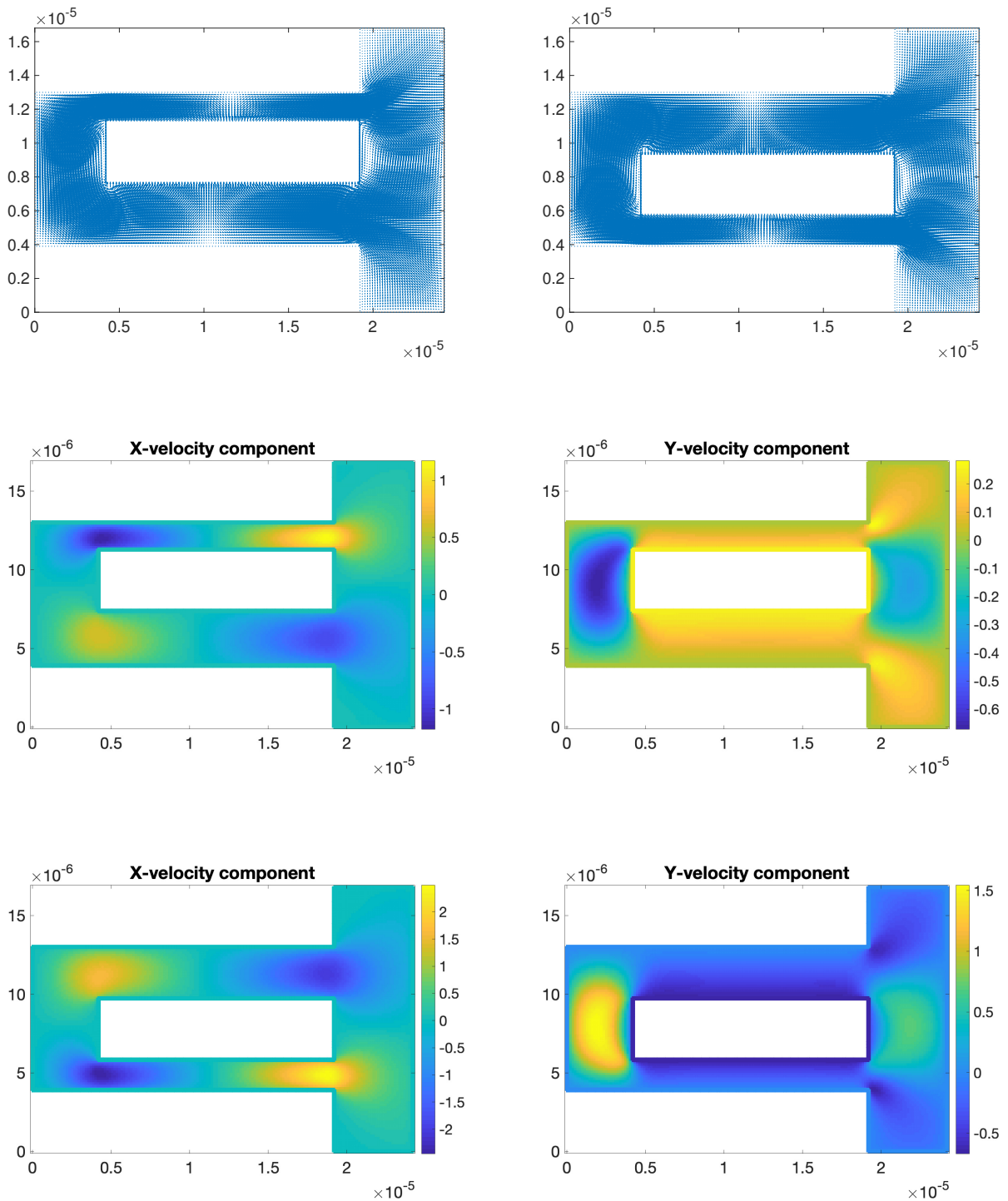


Figure 21. First row: velocity fields at time $t = 1.2 \times 10^{-6}$ and $t = 3 \times 10^{-6}$. Second row: x - and y - velocity components at time $t = 1.2 \times 10^{-6}$. Third row: x - and y - velocity components at time $t = 3.6 \times 10^{-6}$.

4.4.1. Convergence study

In Figure 22 we have plotted the normal stress tensor on the top wall of the shuttle at time $t = 1.2 \times 10^{-6}$. As a reference solution we consider the one obtained at the finest resolution with cell size 4.84×10^{-8} , which corresponds to 111.709 grid points including boundary points. The time step $\Delta t = 3.20 \times 10^{-11}$, which corresponds the CFL number equal to 0.92. For the convergence study we have considered the other coarser grids with sizes 7.5625×10^{-8} , 1.5125×10^{-7} , 3.025×10^{-7} and 6.050×10^{-7} and changed the time steps keeping the constant CFL number equal to 0.92.

In Figure 22 we have plotted the normal stress tensor on the top of the shuttle at time $t = 1.2 \times 10^{-6}$ for different resolutions. In order to estimate the error, we have generated the fixed number of points with $N = 100$ in equal distance. On this grid points we have interpolated the stress tensors from different resolutions including the reference solutions and then defined the relative errors as

$$L_{error}^{rel} = \frac{\sum_{i=1}^N |\phi_{yy,i}^{ref} - \phi_{yy,i}^{\Delta x}|}{\sum_{i=1}^N \phi_{yy,i}^{ref}}. \quad (4.6)$$

We note that $\phi_{yy,i}^{ref}$ is the interpolated reference solution and $\phi_{yy,i}^{\Delta x}$ is the interpolated solution for grid size Δx . In Table 1 we have presented the relative error of the normal stress tensor φ_{yy} at the same time. The errors in the table show the first order convergence of the scheme.

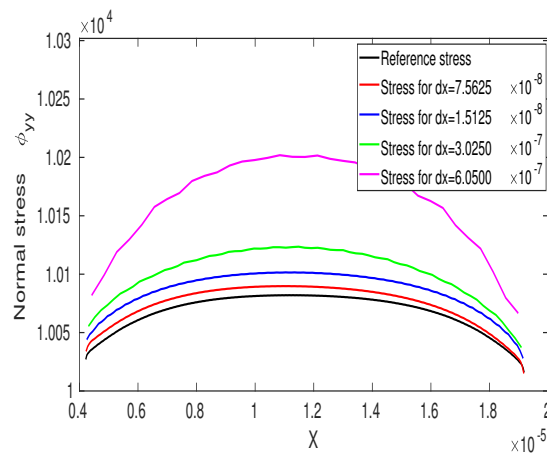


Figure 22. The normal stress tensor on the top wall of the shuttle at $t = 1.2 \times 10^{-6}$ for different cell sizes.

Table 1. Convergence study of the two dimensional moving shuttle at the time $t = 1.2 \times 10^{-6}$.

Δx	Relative error
6.025×10^{-7}	9.3806×10^{-3}
3.025×10^{-7}	3.3410×10^{-3}
1.5125×10^{-7}	1.7921×10^{-3}
7.5625×10^{-8}	6.5300×10^{-4}

4.5. Example 5: Circular body motion in a driven cavity

Consider a circle immersed in a monoatomic gas in a micro square. We consider a 2D spatial and 3D velocity domain. Like in Example 3, the top wall has constant velocity in the positive x-direction. The initial and boundary conditions are the same as in Example 3. Initially, gas and rigid body are at rest. The rigid body is located at the center of the cavity.

We proceed as in Example 2. The force and the torque are computed according to Eq. (2.6). The density of the rigid body is 10 times larger than the density of gas. We have performed the simulations for different density ratios. We experienced that if the density of the rigid body is at least ten times smaller than the density of the gas, instabilities occur in the present set up. A more quantitative comparison of the trajectories for objects with different densities requires a more accurate scheme, and will be performed in a future paper. Again, we use the explicit Euler scheme for the time discretization of the Euler-Newton equations and the same time step for the BGK model. The upper wall moves with velocity $u_w = 30$. We simulate up to the final time $t_{final} = 4.4 \times 10^{-7}$. In Figure 23 we have plotted the path of the center of mass of the body and its positions at different times.

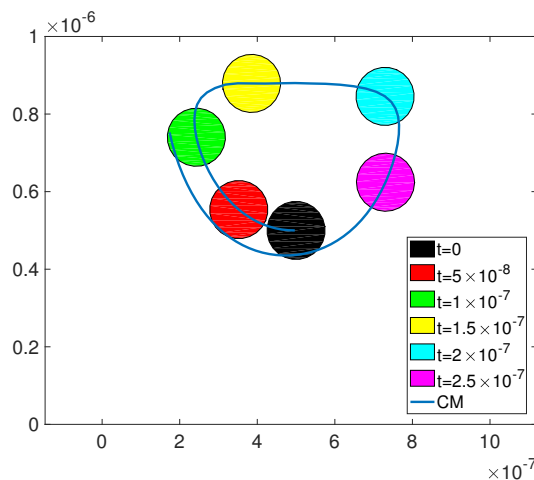


Figure 23. Positions of circular particle at time $t = 0, 5 \times 10^{-8}, 1 \times 10^{-7}, 1.5 \times 10^{-7}, 2 \times 10^{-7}$ and 2.5×10^{-7} (clockwise direction) together with the trajectory of the center of mass.

Since there are no analytical or experimental results to validate the numerical solutions, we validate our solutions with DSMC simulations for the Boltzmann equation. We use the same initial and boundary conditions and the same parameters for both schemes. First, we consider $u_w = 10, 20$ and 30 for the DSMC simulation. As we have seen in earlier examples, the DSMC results are dominated from the statistical noise for smaller Mach number flows. In Figure 24 we have compared the trajectories of the center of mass obtained from both methods. We observe that increasing the wall velocity u_w gives a better agreement between the numerical solution of the BGK model and the DSMC solution. In Figure 24 we have plotted the center of mass obtained from the BGK model and DSMC simulations. In the case of DSMC simulations 10 independent runs are carried out. For larger u_w the BGK solutions and DSMC solutions are getting closer.

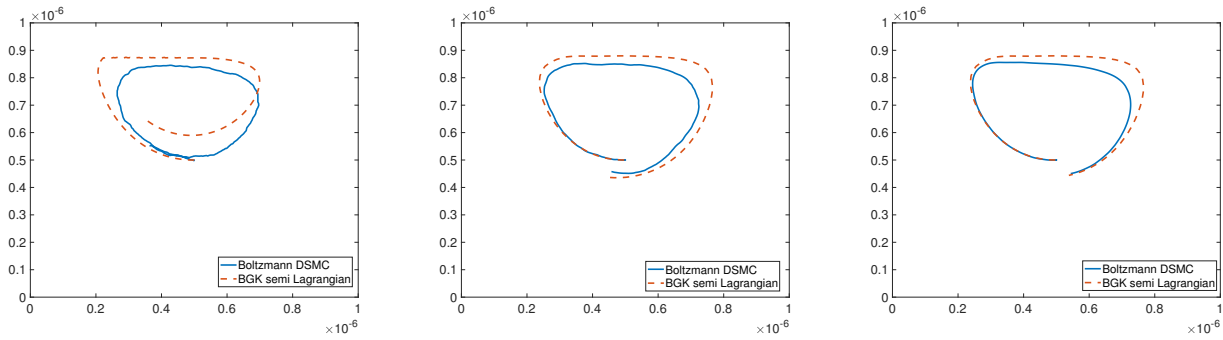


Figure 24. Comparison of the trajectories of the center of mass obtained from the BGK model and DSMC simulations for $u = 10$ (left), 20 (middle) and 30 (right).

4.6. *Example 6: Rigid body with arbitrary shapes in a driven cavity*

To show that the scheme is able to simulate the interaction of the gas with an arbitrary shaped rigid body, we have considered a 2D spatial and 2D velocity domain and three different types of bodies, which are triangular, L-shaped and chiral particles. For these three shapes, rotational effects are clearly observed. The initial and boundary conditions are as in Example 5. The density of the rigid bodies is again 10 times larger than the density of the gas. The upper wall has velocity $u_w = 30$ and the simulations are stopped after time 4.4×10^{-7} for all cases. All rigid bodies are initialised at the center of the cavity. In Figures 25–27 we have plotted the positions at different times together with the trajectories of triangle, L-shaped and chiral particles, respectively. In all cases we see that the rigid bodies follow the flow path. We mention here that in [24] a general method for the simulation of arbitrary shaped object in a rarefied gas has been presented. In that paper the gas satisfies the Boltzmann transport equation, which is effectively solved by DSMC.

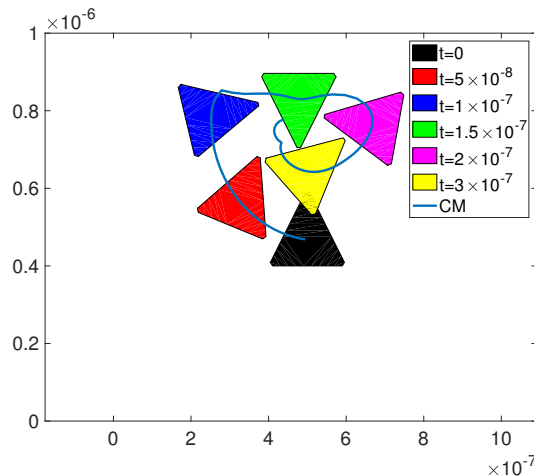


Figure 25. Positions of triangular particle at time $t = 0, 5 \times 10^{-8}, 1 \times 10^{-7}, 1.5 \times 10^{-7}, 2 \times 10^{-7}, 2 \times 10^{-7}$ and 3×10^{-7} (clockwise direction) together with the trajectory of center of mass.

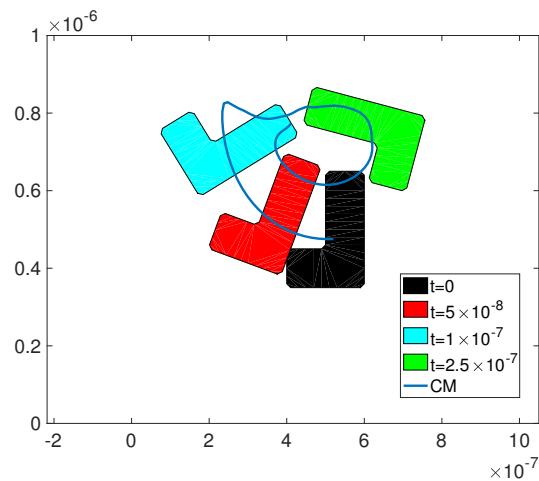


Figure 26. Positions of L-shaped particle at time $t = 0, 5 \times 10^{-8}, 1 \times 10^{-7}$ and 2.5×10^{-7} (clockwise direction) together with the trajectory of center of mass.

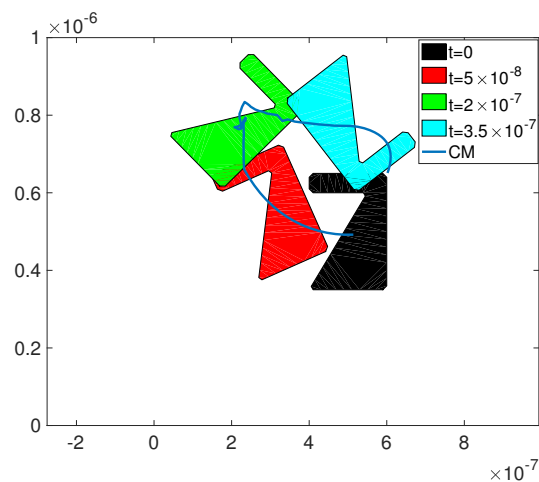


Figure 27. Positions of chiral particle at time $t = 0, 5 \times 10^{-8}, 2 \times 10^{-7}$ and 3.5×10^{-7} together with the trajectory of center of mass.

5. Conclusions and Outlook

In this paper, we have presented a mesh free method for the simulation of moving rigid bodies immersed in a rarefied gas flow. The motion of the rigid body is obtained by solving the Newton-Euler equations. The force and the torque are computed from the surrounding gas. The Newton-Euler equations are solved by an explicit Euler method. The rarefied gas is simulated by solving the BGK model of the Boltzmann equation. A semi-Lagrangian method is used to solve the BGK model, where a first order least squares approximation is used for the interpolation scheme. Several numerical tests are performed in order to validate the method, both in one and two space dimensions. In particular, in

1D we consider the case of a moving plate immersed in a rarefied gas. In a first test we assume the motion of the plate is prescribed (one way coupling), while in a second test the motion of the plate is computed from Newton's equations (two way coupling). In both cases we compared the results with those obtained by DSMC solution of the Boltzmann equation. Notice that DSMC results required to take the average of a lot of runs in order to decrease statistical fluctuations. In two space dimensions we considered several test problems: some in which the motion of the object is prescribed, such as the classical driven cavity (and compared the results with DSMC) and the motion of the shuttle in a 2D model of a Micro Electro Mechanical System (and results are compared with others available in the literature [13]). Finally, some tests are performed with a rigid body of arbitrary shape immersed in a gas and driven by the flow (two way coupling). In some cases the results are compared with those obtained by DSMC. In the regimes we investigated there is a good qualitative agreement between the solutions obtained by BGK and by the full Boltzmann equation simulated by DSMC. Of course accurate DSMC solutions require a computational time which is several orders of magnitude higher than the one needed by the numerical solution of the BGK model.

In this paper we consider a one way heat exchange: the temperature of the rigid body is assumed to be constant in space and time, which is equivalent to suppose that the heat capacity of the rigid body is much larger than the one of the gas. In future work we shall remove such an approximation and consider finite heat capacity of the rigid body. As a first step in this direction we assume a rigid body with infinite conductivity, which will make the temperature of the body constant in space. Later on we shall model heat diffusion in the solid as well.

Moreover, the scheme will be extended to the case of gas-mixtures [15] and to three space dimensions. An interesting application of the method will include the treatment of several bodies immersed in a rarefied gas. In this way it will be possible to model a collection mesoscopic particles dispersed in a rarefied gas, thus providing a quantitative tool that can be used to validate homogenised macroscopic models of suspensions.

From the methodological point of view, further research directions will include the use of non-oscillatory higher order methods in space and time based on least squares approaches, see [1] for a combination of WENO and least squares approaches for fluid dynamic equations and high order approximation of boundary conditions.

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Conflict of interest

The authors declare no conflict of interest.

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