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*Research article*

## **Adaptive nonsingular terminal sliding mode controller for PMSM drive system using modified extended state observer**

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**Abstract:** In this study, a novel adaptive nonsingular terminal sliding mode (ANTSM) control frame combined with a modified extended state observer (MESO) is presented to enhance the anti-interference performance of the permanent magnet synchronous motor (PMSM) system. In the face of time-varying disturbances with unknown upper bounds, traditional nonsingular terminal sliding mode (NTSM) controllers typically utilize the large control gain to counteract the total disturbance, which will cause unsatisfactory control performances. To address this tough issue, an ANTSM control technique was constructed for the PMSM system by tuning the control gain automatically without overestimation. On this basis, the MESO was adopted to estimate the unknown total disturbance, whose estimation was offset to the ANTSM control input. By applying finite-time techniques, the estimation error will finite-time converge to zero. The proposed MESO has a more rapid estimation speed than the traditional extended state observer (ESO). Finally, the validity of the ANTSM + ESO composite control algorithm is confirmed by comprehensive experiments.

**Keywords:** PMSM; disturbance observer; terminal sliding mode; composite control; adaptive law

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### **1. Introduction**

Owing to the distinctive characteristics of high power density and high efficiency, permanent magnet synchronous motor (PMSM) has drawn extensive popularity over the last several decades [1–3]. For this reason, PMSM has been widely applied for industry, especially for electric vehicles, vacuum cleaners, electric ship propulsion and more. Nevertheless, the traditional proportional integral (PI) control algorithm is typically difficult to offer the high-precision control property [4–7]. To improve the unsatisfactory performance of the PI controller, lots of advanced algorithms have been applied to further optimize the control precision, such as predictive control [8,9], fuzzy control [10–13], adaptive

control [14–18] and sliding mode control (SMC) [19–23].

Among the previously mentioned modern control approaches, the SMC is an effective strategy to enhance the anti-disturbance capability of the PMSM system [24]. A traditional linear SMC method was presented in [25] to obtain a fast dynamic response and strong robustness. However, the parameter variation has not been completely considered in theory, which limits the stable operation of motors. In [26], a new piecewise but differentiable switching controller was introduced with a proper design idea to avert the singularity problem. Moreover, in order to make the speed error converge faster, the nonsingular terminal sliding mode (NTSM) control algorithm was proposed [27] to make sure that the speed error will finite-time converge to the origin.

The NTSM strategy can handle the singular issue and provide satisfactory tracking accuracy. However, the NTSM controller still brings severe chattering [28, 29], and a fast NTSM controller with no switching function was used to weaken the chattering [30]. However, this kind of controller has poor anti-disturbance performance and only converges the speed error to a region instead of to zero. At the same time, a brand-new adaptive terminal sliding mode (TSM) reaching law combined with a fast TSM control method was utilized to construct the speed controller for the PMSM system in [31]. Even if an adaptive law was adopted to reduce chattering, this method still cannot adjust the control gain automatically with the change of total disturbance.

Notably, utilizing the disturbance observer (DOB) to estimate the unknown time-varying disturbance is a valid strategy to avoid the overestimated switching gain in SMC [32–34]. Compensating the precise estimates to the baseline controller, the PMSM drive system will achieve significant steady-state performance and strong disturbance rejection property synchronously [35]. Without the accurate information of the PMSM mathematical model, a conventional extended state observer (ESO) was applied [36] to restrain the property deterioration of PMSM in the existence of external disturbance and parameter variation. Unfortunately, the baseline controller only adopted the linear PI method. It is extremely hard for this category of composite controllers to achieve strong disturbance rejection property. On this basis, the traditional ESO combined with a simple SMC algorithm was constructed [37] to address the above tough issue. Although the traditional ESO can estimate the total disturbance exactly, it just guarantees that the estimated error converges asymptotically to zero, which may result in poor estimation precision. In order to obtain higher estimation accuracy and robustness, various improved ESOs have been proposed in recent years [38, 39]. In [40], the natural evolution theory has been applied to graph structure learning, where an evolutionary method was constructed to evolve a population of graph neural network (GNN) models to adapt to dynamical environments. In [41], a heterogeneous network representation learning method was reported to characterize implicitly inside Ethereum transactions. In [42], a center-based transfer feature learning with classifier adaptation for the surface defect recognition was proposed. In [43], the authors proposed an integrated triboelectric nanogenerator and tribovoltaic nanogenerator in the air cylinder as difunctional pneumatic sensors for simultaneous position and velocity monitoring. In [44], the recent developments in the area of arc fault detection were studied.

In this paper, a novel adaptive nonsingular terminal sliding mode (ANTSM) controller combined with a modified ESO (MESO) is proposed to enhance the disturbance rejection property of the PMSM system. First of all, considering the parameter variation and time-varying load torque, an ANTSM controller is constructed to provide the desired steady-state and dynamic performance. Furthermore, in order to solve the trouble of unsatisfactory control effect due to large switching gain, one novel

MESO is utilized to observe the unknown time-varying disturbance and compensate the estimates to the ANTSM controller simultaneously. Lastly, the validity of the ANTSM + ESO composite control algorithm is proved by comprehensive experiments. The main contribution of this article could be summarized in the following points:

- 1) A novel NTSM controller is designed. The control gain is automatically tuned by the proposed adaptive law. It avoids unsatisfactory control performance caused by excessive control gain.
- 2) A MESO is proposed to improve the anti-disturbance capability of the PMSM system. By using the finite-time technique, the estimation error can converge to zero in finite time. The proposed MESO has a higher estimation accuracy and faster estimation speed.
- 3) The proposed composite controller combines the ANTSM algorithm with MESO to improve the system robustness. Since the high disturbance estimation accuracy of MESO, the rejection ability to disturbances of the PMSM speed regulation system can be improved.

The rest of the paper is organized as follows. In Section 2, the PMSM mathematical model with parameter perturbation and the traditional NTSM control method are described. Section 3 shows the design of the proposed MESO-based ANTSM controller in detail. Comprehensive experiments are illustrated in Section 4. The summary of this paper is presented in Section 5.

**Notations:** Throughout the paper, the symbol  $\lfloor x \rfloor^m$  is used to present the  $|x|^m \cdot \text{sign}(x)$  for a real number  $m$ , and the symbol  $\lfloor x \rfloor^*$  is defined as

$$\lfloor x \rfloor^* = \begin{cases} 1, & x \geq 0 \\ 0, & x < 0 \end{cases}.$$

## 2. Problem formulation

### 2.1. Mathematical model of PMSM

In the ideal case, the motion equation of PMSM can be given by

$$\dot{\Omega} = \frac{1.5n_p\psi_f}{J}i_q - \frac{B}{J}\Omega - \frac{T_L}{J} \quad (2.1)$$

where  $\Omega$  is the rotor angular speed,  $n_p$  is the number of pole pairs,  $\psi_f$  is the flux linkage,  $J$  is the moment of inertia,  $B$  is the viscous friction coefficient,  $i_q$  is the stator current in the  $q$ -axis and  $T_L$  is the load torque.

Because of load variation in practical applications, the value of inertia may be mismatched. Therefore, we introduce  $\Delta J = J - J_0$ , where  $J_0$  is the nominal value and  $\Delta J$  is parameter variation. Then, replacing current  $i_q$  by the reference current  $i_q^*$ , system (1) can be rewritten as

$$\begin{aligned} \dot{\Omega} &= \frac{1.5n_p\psi_f}{J_0}i_q - \frac{B}{J_0}\Omega + d_0(t) \\ &= bi_q^* - \frac{B}{J_0}\Omega + d_0(t) \end{aligned} \quad (2.2)$$

where  $d_0(t) = -\frac{T_L}{J_0} + b(i_q - i_q^*) - \frac{\Delta J}{J_0}\dot{\Omega}$  and  $b = \frac{1.5n_p\psi_f}{J_0}$ .

## 2.2. Traditional NTSM controller design

The traditional NTSM algorithm is used to design the speed controller for the PMSM system in this subsection. First of all, setting  $\Omega_r$  as the reference speed, the speed error  $\Omega_e$  is expressed as

$$\Omega_e = \Omega_r - \Omega. \quad (2.3)$$

With the help of (2.2), one has

$$\dot{\Omega}_e = -bi_q^* - \frac{B}{J_0}\Omega_e + d(t) \quad (2.4)$$

where  $d(t) = -d_0(t) + \dot{\Omega}_r + \frac{B}{J_0}\Omega_r$  is the total disturbance.

**Assumption 1:** [45–47] *The total disturbance  $d(t)$  is bounded and differentiable, and there exist known positive constants  $l_1$  and  $l_2$  such that  $|d(t)| \leq l_1$  and  $|\dot{d}(t)| \leq l_2$ .*

**Remark 1:** From (2.2) and (2.4), we can get that the disturbance  $d(t)$  consists of the load torque  $T_L$ , the stator current  $i_q$ , the stator reference current  $i_q^*$ , the speed  $\Omega$ , the setting speed  $\Omega_r$  and other components. It can be concluded that the above variables are bounded and differentiable. Therefore, it is reasonable that the total disturbance satisfies  $|d(t)| \leq l_1$  and  $|\dot{d}(t)| \leq l_2$ , where  $l_1$  and  $l_2$  are known positive constants.

According to [48], the nonsingular terminal sliding manifold is selected as

$$s = \int \Omega_e dt + \frac{1}{\beta}\Omega_e^{p/q} \quad (2.5)$$

where  $\beta$  is positive constant,  $p$  and  $q$  are positive odd integers and  $1 < p/q < 2$ .

Based on this, the NTSM controller will be designed as

$$i_q^* = \frac{1}{b} \left( -\frac{B}{J_0}\Omega_e + \beta \frac{q}{p}\Omega_e^{2-p/q} + k \cdot \text{sign}(s) \right) \quad (2.6)$$

with a positive constant  $k$ .

Choose the widely-used Lyapunov function as

$$V(s) = \frac{1}{2}s^2. \quad (2.7)$$

Differentiating  $V(s)$  gets

$$\begin{aligned} \dot{V}(s) &= s\dot{s} \\ &= s \left( \Omega_e + \frac{p}{\beta q}\Omega_e^{p/q-1}\dot{\Omega}_e \right) \\ &= s \frac{p}{\beta q}\Omega_e^{p/q-1} \left( -bi_q^* - \frac{B}{J_0}\Omega_e + d(t) + \beta \frac{q}{p}\Omega_e^{2-p/q} \right). \end{aligned} \quad (2.8)$$

Substituting NTSM controller (2.6) into (2.8), one obtains

$$\dot{V}(s) = s \frac{p}{\beta q}\Omega_e^{p/q-1}(-k \cdot \text{sign}(s) + d(t))$$

$$\begin{aligned}
&\leq -\frac{P}{\beta q} \Omega_e^{p/q-1} |s| k + \frac{P}{\beta q} x^{p/q-1} |s| |d(t)| \\
&\leq -\frac{P}{\beta q} \Omega_e^{p/q-1} |s| (k - l_1). \tag{2.9}
\end{aligned}$$

Similar to [27], if the control gain satisfies  $k > l_1$ , then the speed error will converge to origin in a finite time.

**Remark 2:** To guarantee the stability of the PMSM, the value of  $k$  in the traditional NTSM controller should be chosen to be larger than the upper bound of the total disturbance. However, the total disturbance in practical engineering is time-varying, and its upper bound may be much smaller than the fixed control gain. Since the control gain directly affects the chattering amplitude, conservative control gain may lead to unsatisfactory control performance in PMSM. Thus, it is urgent to design a novel controller with variable and a small enough control gain to suppress the time-varying disturbance.

### 3. Proposed controller design

To address the forgoing disadvantages of the conventional NTSM controller, a novel ANTSM is developed in this section to automatically tune the value of control gains. Based upon this, compensating the disturbance estimation value derived by MESO to the ANTSM controller, the control gain can get a further reduction.

#### 3.1. Construction of ANTSM controller

With the aid of controller (2.6), the ANTSM controller is constructed as

$$i_q^* = \frac{1}{b} \left( -\frac{B}{J_0} \Omega_e + \beta \frac{q}{p} \Omega_e^{2-p/q} + k(t) \cdot \text{sign}(s) \right). \tag{3.1}$$

The adaptive law  $k(t)$  is given as

$$\begin{cases} \dot{k}(t) = \eta \cdot k(t) \cdot \text{sign}(\delta(t)) + N[k_M - k(t)]^* \\ \quad + N[k_m - k(t)]^* \\ \delta(t) = |[\text{sign}(s)]_{av}| - \varepsilon, \end{cases} \tag{3.2}$$

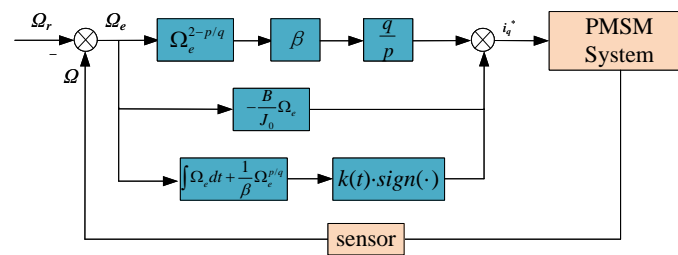
where  $k_M > l_1$  and  $k_m > 0$  are the maximums and minimums of the control gain, constant  $\varepsilon \in (0, 1)$ ,  $\eta > \frac{l_2}{\varepsilon k_m}$ ,  $N > \eta k_M$ , and  $[\text{sign}(s)]_{av}$  is produced by the signal  $z(t)$  of the low-pass filter

$$\dot{z} = \frac{1}{\lambda} (\text{sign}(s) - z), \quad z(0) = 0 \tag{3.3}$$

with  $\lambda$  being the tunable parameter. The ANTSM controller structure is depicted by Figure 1.

**Remark 3:** Since the discontinuous switching function  $\text{sign}(s)$  varies at negative one and one, the function  $[\text{sign}(s)]_{av}$  is continuous and its value belongs to  $(-1, 1)$ . Therefore, it can be assumed that the second derivative of function  $[\text{sign}(s)]_{av}$  satisfies  $|\frac{d^2}{dt^2} [\text{sign}(s)]_{av}| \leq C$  and  $C > 0$ .

Next, we will prove that  $k(t)$  can converge in a finite time to the minimum absolute value of the total uncertainty. In addition, the equivalent control theory is crucial for the proof in the paper.



**Figure 1.** The diagram of proposed ANTSM controller.

The equivalent controller is constructed as

$$u_{eq} = \frac{1}{b} \left( -\frac{B}{J_0} \Omega_e + \beta \frac{q}{p} \Omega_e^{2-p/q} + d(t) \right). \tag{3.4}$$

Since the exact information about the total disturbance  $d(t)$  is not available, the equivalent controller (3.4) cannot be directly applied to the PMSM system. Therefore, an average control of ANTSM controller (3.1) is employed to track controller (3.4). Meanwhile, to enhance the accuracy of the average control, a continuous function  $[\text{sign}(s)]_{av}$  is used to replace the discontinuous function  $\text{sign}(s)$  in equivalent controller. Then, the average control of ANTSM controller (3.1) is constructed as

$$u_{av} = \frac{1}{b} \left( -\frac{B}{J_0} \Omega_e + \beta \frac{q}{p} \Omega_e^{2-p/q} + k(t) \cdot [\text{sign}(s)]_{av} \right). \tag{3.5}$$

According to the above analysis, if the average control (3.5) can track the equivalent controller (3.4) well, one has

$$d(t) = k(t) \cdot [\text{sign}(s)]_{av}. \tag{3.6}$$

If  $[\text{sign}(s)]_{av}$  approaches one, the  $k(t)$  is sufficiently large to counteract the  $d(t)$ . The basic idea of the ANTSM control approach is to ensure that  $k(t)$  can converge to  $\frac{|d(t)|}{|[\text{sign}(s)]_{av}|}$  with  $\varepsilon$  being close to one in a finite time. Meanwhile, the continuous function  $[\text{sign}(s)]_{av}$  should converge to parameter  $\varepsilon$ .

Consider the following Lyapunov function:

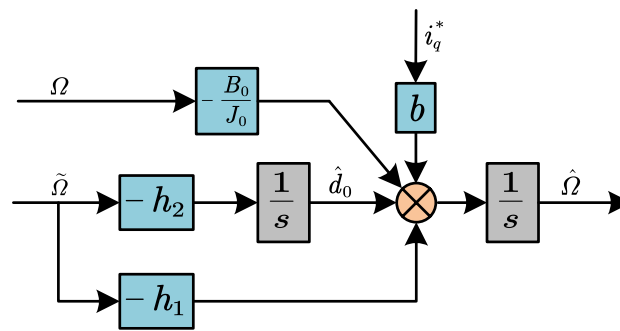
$$V_1(\delta) = \frac{1}{2} \delta^2. \tag{3.7}$$

Evaluating the derivative of  $V_1(\delta)$  yields

$$\dot{V}_1(\delta) = \delta \dot{\delta} = \delta \frac{d}{dt} \left( \frac{|d(t)|}{k(t)} \right). \tag{3.8}$$

Since the range of control gain  $k(t) \in [k_m, k_M]$ , one has  $\frac{|d(t)|}{\varepsilon} > k_m$ . With the help of Assumption 1, (3.8) can be rewritten as

$$\dot{V}_1(\delta) \leq -|\delta| k^{-1}(t) (\varepsilon k_m \eta - l_2). \tag{3.9}$$



**Figure 2.** Diagram of traditional ESO.

Combining  $\eta > \frac{l_2}{\varepsilon k_m}$  and (3.9), one has

$$\dot{V}_1(\delta) \leq -\frac{|\delta|(\varepsilon k_m \eta - l_2)}{k_M} = -\kappa V_1^{\frac{1}{2}}(\delta) \quad (3.10)$$

where  $\kappa = \frac{\sqrt{2}(\varepsilon k_m \eta - l_2)}{k_M} > 0$ .

Therefore, inequality (3.10) satisfies the finite-time stability theorem [49], and it indicates that  $k(t)$  will converge to  $\frac{|d(t)|}{\varepsilon}$ .

### 3.2. Construction of MESO

In this subsection, the conventional ESO and the presented MESO are exploited to estimate the disturbance  $d_0(t)$ , respectively. First, provided that  $|d_0| \leq l_0, l_0 > 0$ , the traditional ESO designed for system (2.2) can be expressed as

$$\begin{cases} \dot{\hat{\Omega}} = \hat{d}_0 - \frac{B}{J_0} \Omega + b i_q^* - h_1 \tilde{\Omega}, \\ \dot{\hat{d}}_0 = -h_2 \tilde{\Omega} \end{cases} \quad (3.11)$$

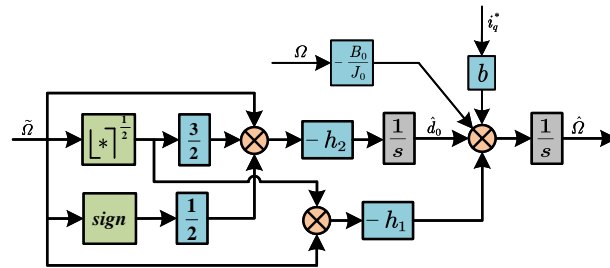
where  $\tilde{\Omega} = \hat{\Omega} - \Omega$ ,  $\hat{\Omega}$  and  $\hat{d}_0$  are the estimated values of  $\Omega$  and  $d_0$ ,  $h_1$  and  $h_2$  are the gains of traditional ESO.

Nevertheless, the traditional ESO just ensures that the estimation of error converges to zero asymptotically, which causes poor estimation precision. Therefore, it is urgent to accelerate the estimation speed of traditional ESO. For (2.2), the MESO is constructed as

$$\begin{cases} \dot{\hat{\Omega}} = \hat{d}_0 - \frac{B}{J_0} \Omega + b i_q^* - h_1 \phi_1(\tilde{\Omega}), \\ \dot{\hat{d}}_0 = -h_2 \phi_2(\tilde{\Omega}) \end{cases} \quad (3.12)$$

where functions  $\phi_1(\tilde{\Omega})$  and  $\phi_2(\tilde{\Omega})$  are given by

$$\begin{aligned} \phi_1(\tilde{\Omega}) &= \left[ |\tilde{\Omega}|^{\frac{1}{2}} + \tilde{\Omega} \right], \\ \phi_2(\tilde{\Omega}) &= \frac{1}{2} \text{sign}(\tilde{\Omega}) + \frac{3}{2} \left[ |\tilde{\Omega}|^{\frac{1}{2}} + \tilde{\Omega} \right]. \end{aligned} \quad (3.13)$$



**Figure 3.** Diagram of proposed MESO.

Subtracting (3.12) from (2.2) obtains

$$\begin{cases} \dot{\tilde{\Omega}} = \tilde{d}_0 - h_1 \phi_1(\tilde{\Omega}), \\ \dot{\tilde{d}}_0 = -h_2 \phi_2(\tilde{\Omega}) - \dot{d}_0 \end{cases} \quad (3.14)$$

where  $\tilde{d}_0 = \hat{d}_0 - d_0$ .

By defining  $\xi^T = [\phi_1(\tilde{\Omega}), \tilde{d}_0]$ , the time derivative of  $\xi$  can be written as

$$\begin{aligned} \dot{\xi} &= \Phi(\tilde{\Omega}) \begin{bmatrix} -h_1 \phi_1(\tilde{\Omega}) + \tilde{d}_0 \\ -h_2 \phi_1(\tilde{\Omega}) - \frac{\dot{d}_0}{\Phi(\tilde{\Omega})} \end{bmatrix} \\ &= \Phi(\tilde{\Omega}) (A\xi - B\psi) \end{aligned} \quad (3.15)$$

where  $A = \begin{bmatrix} -h_1 & 1 \\ -h_2 & 0 \end{bmatrix}$ ,  $B = \begin{bmatrix} 0 \\ 1 \end{bmatrix}$ ,  $\Phi(\tilde{\Omega}) = \frac{3}{2} |\tilde{\Omega}|^{\frac{1}{2}} + 1$  and  $\psi = \dot{d}_0 / \Phi(\tilde{\Omega})$ .

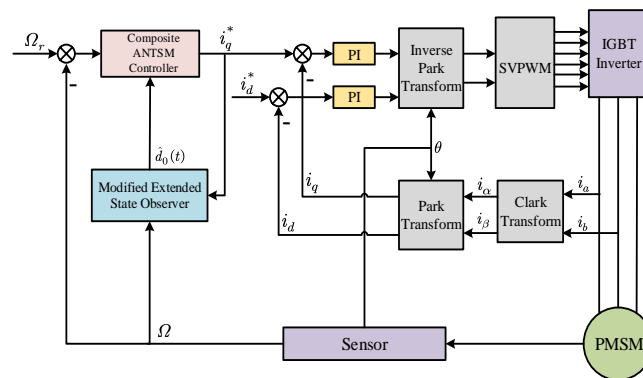
With the help of  $|\dot{d}_0| \leq l_0$ , one yields  $|\psi| \leq l_0$ . Then, we define

$$\begin{aligned} \Gamma(\psi, \xi) &= \begin{bmatrix} \xi \\ \psi \end{bmatrix}^T \begin{bmatrix} l_0^2 & 0 \\ 0 & -1 \end{bmatrix} \begin{bmatrix} \xi \\ \psi \end{bmatrix} \\ &= \begin{bmatrix} \phi_1(\tilde{\Omega}) & \tilde{d}_0 & \psi \end{bmatrix} \begin{bmatrix} l_0^2 & 0 \\ 0 & -1 \end{bmatrix} \begin{bmatrix} \phi_1(\tilde{\Omega}) \\ \tilde{d}_0 \\ \psi \end{bmatrix} \\ &= \begin{bmatrix} \xi^T & \psi \end{bmatrix} \begin{bmatrix} l_0^2 & 0 \\ 0 & -1 \end{bmatrix} \begin{bmatrix} \xi \\ \psi \end{bmatrix} \\ &= -\psi^2 + l_0^2 \\ &\geq 0. \end{aligned} \quad (3.16)$$

**Theorem 1:** For a positive constant  $\gamma$  and the symmetric and positive definite matrix  $Q$ , if the following inequality holds

$$\begin{bmatrix} A^T Q + QA + \gamma Q + l_0^2 & QB \\ B^T Q & -1 \end{bmatrix} \leq 0,$$





**Figure 4.** Control scheme of proposed ANTSM+MESO controller.

then the estimation error of the proposed MESO (3.12) will converge to zero in a finite time.

**Proof.** The Lyapunov function is chosen as

$$V_2 = \xi^T Q \xi. \tag{3.17}$$

From (3.15) and (3.16), the differential coefficient of  $V_2$  can be given as

$$\begin{aligned} \dot{V}_2 &= \Phi(\tilde{\Omega}) \left[ \xi^T (A^T Q + PA) \xi + \psi B^T Q \xi + \xi^T QB \psi \right] \\ &= \Phi(\tilde{\Omega}) \begin{bmatrix} \xi \\ \psi \end{bmatrix}^T \begin{bmatrix} A^T Q + QA & QB \\ B^T Q & 0 \end{bmatrix} \begin{bmatrix} \xi \\ \psi \end{bmatrix} \\ &\leq \Phi(\tilde{\Omega}) \left\{ \begin{bmatrix} \xi \\ \psi \end{bmatrix}^T \begin{bmatrix} A^T Q + QA & QB \\ B^T Q & 0 \end{bmatrix} \begin{bmatrix} \xi \\ \psi \end{bmatrix} + \Gamma(\psi, \xi) \right\} \\ &\leq \Phi(\tilde{\Omega}) \begin{bmatrix} \xi \\ \psi \end{bmatrix}^T \begin{bmatrix} -\gamma Q & 0 \\ 0 & 0 \end{bmatrix} \begin{bmatrix} \xi \\ \psi \end{bmatrix} \\ &= \Phi(\tilde{\Omega}) (-\gamma \xi^T Q \xi) \\ &= -\Phi(\tilde{\Omega}) \gamma V_2 \\ &= -\frac{1}{2|\tilde{\Omega}|^{\frac{1}{2}}} \gamma V_2 - \gamma V_2. \end{aligned} \tag{3.18}$$

From (3.17), it is derived that

$$\lambda_{\min}\{Q\} \|\xi\|_2^2 \leq \xi^T Q \xi \leq \lambda_{\max}\{Q\} \|\xi\|_2^2 \tag{3.19}$$

where  $\lambda\{\cdot\}$  is the eigenvalue of matrix  $\{\cdot\}$  and  $\|\xi\|_2^2$  is the Euclidean norm of  $\xi$ .

Considering  $\|\xi\|_2^2 = |\tilde{\Omega}| + 2|\tilde{\Omega}|^{\frac{3}{2}} + \tilde{\Omega}^2 + \tilde{d}_0^2$ , it follows that

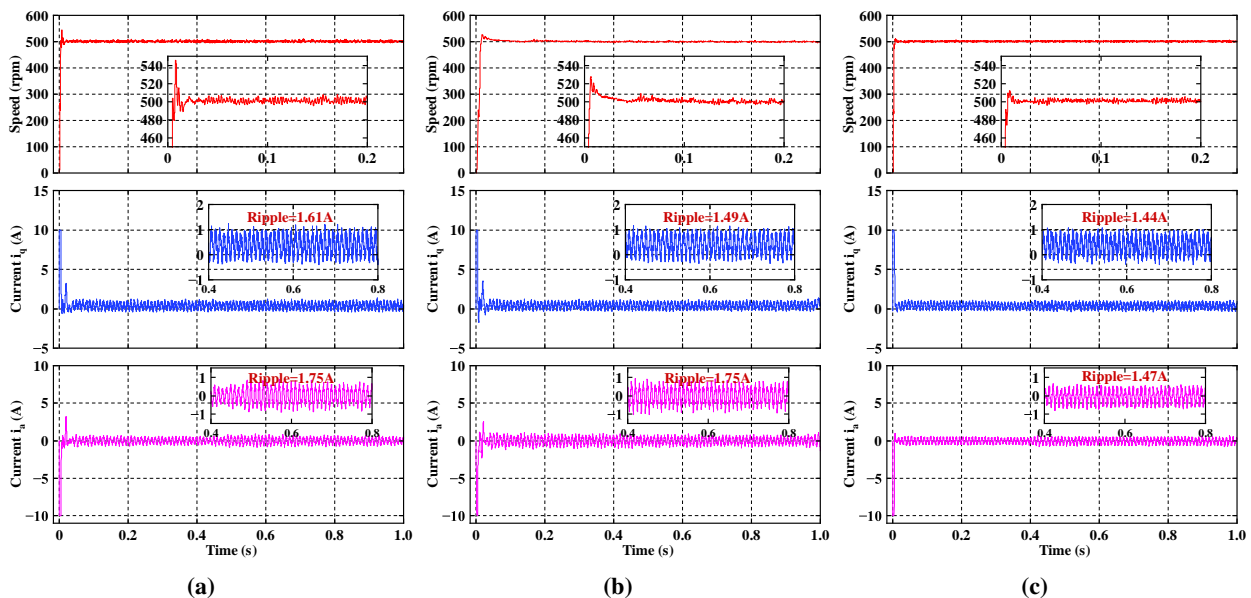
$$|\tilde{\Omega}|^{\frac{1}{2}} \leq \|\xi\|_2 \leq \frac{V_2^{\frac{1}{2}}}{\lambda_{\min}^{\frac{1}{2}}\{P\}}. \tag{3.20}$$

In accordance with (3.18) and (3.20), one obtains

$$\dot{V}_2 \leq -\frac{1}{2|\tilde{\Omega}|^{\frac{1}{2}}} \gamma V_2 - \gamma V_2$$

**Table 1.** Parameters of the PMSM.

Name	Value and unit
dc-bus voltage	220 V
Rated torque	10 N · m
Machine pole pairs	4
Rated speed	1500 rpm
Rotor flux linkage	0.142 wb
Moment of inertia	1.94 Kg/m <sup>2</sup>
Rated power	1.5 kW
Stator resistance	1.5 Ω
Sampling frequency	10 kHz

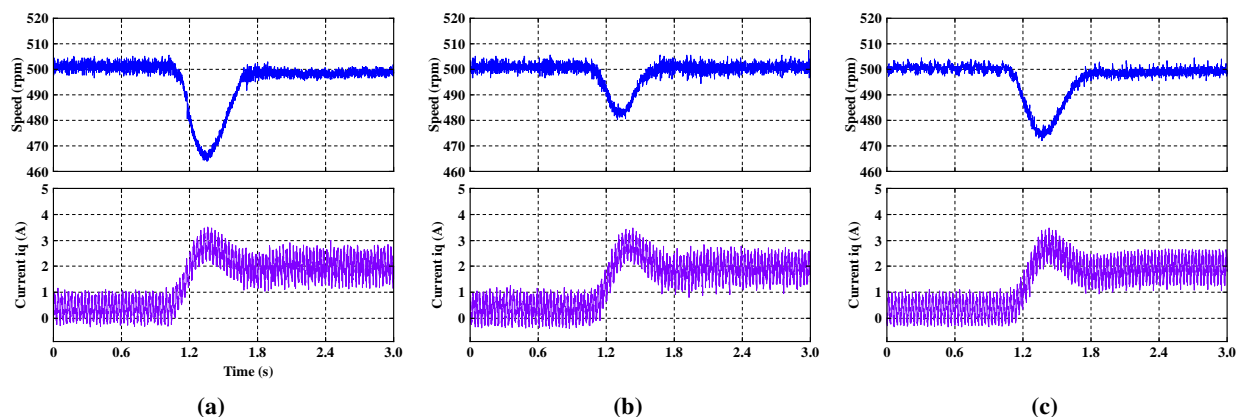
**Figure 5.** Response curve under the step change of speed in the start-up phase. (a) PI. (b) NTSM (c) ANTSM.

$$\leq -\frac{\gamma \lambda^{\frac{1}{2}} \min\{P\}}{2} V_2^{\frac{1}{2}} - \gamma V_2. \quad (3.21)$$

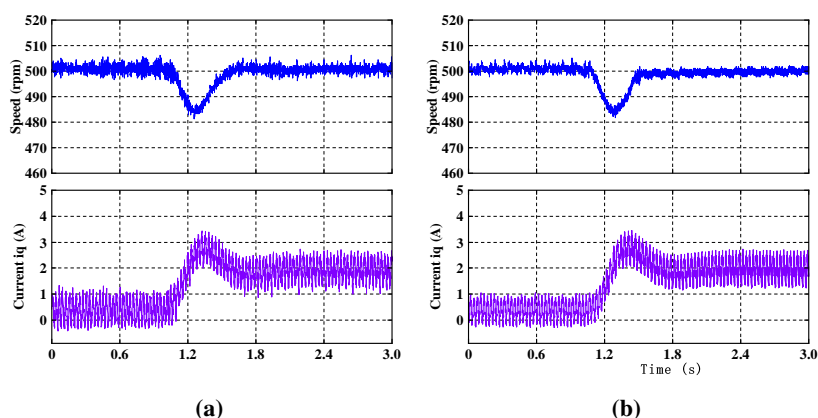
Clearly, (3.21) satisfies the finite-time stability theory [50]. To this end, we have proved that under the proposed MESO (3.12) the estimation error will be guaranteed to converge to zero in a finite time.

The proposed MESO can estimate the disturbance precisely. Then, compensating the estimate  $\hat{d}_0(t)$  to the baseline controller, the output signal of the MESO-based ANTSM controller can be designed as

$$i_q^* = \frac{1}{b} \left( -\frac{B}{J_0} \Omega_e + \beta \frac{q}{p} x^{2-p/q} + k(t) \cdot \text{sign}(s) - \hat{d}_0(t) \right). \quad (3.22)$$



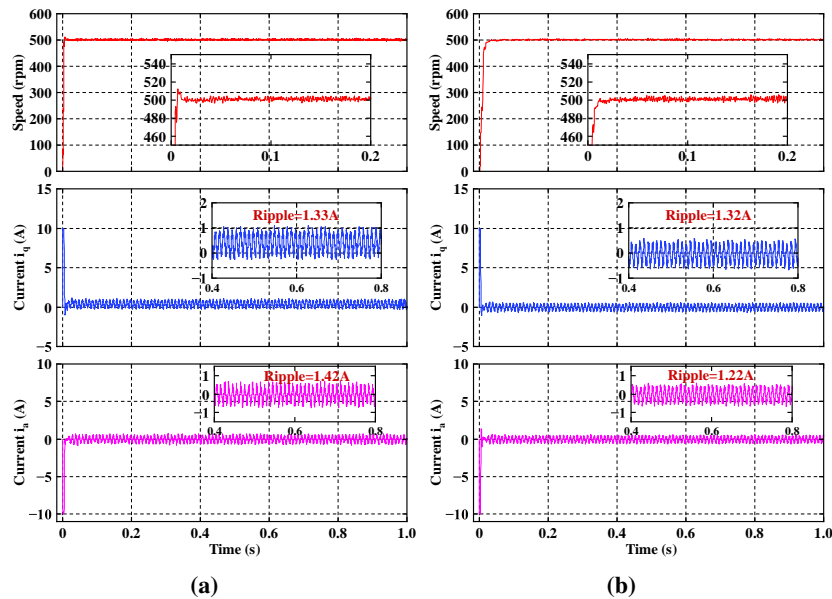
**Figure 6.** Response curves under a sudden load torque change. (a) PI. (b) NTSM. (c) ANTSM.



**Figure 7.** Response curves under a sudden load torque change. (a) ANTSM+ESO. (b) ANTSM + MESO.

#### 4. Experimental validation

Aiming to validate the performance of the ANTSM + MESO control algorithm, comprehensive experimental results are given in this section. The complete schematic of the proposed ANTSM + MESO control system is illustrated in Figure 4. The experimental platform mainly consists of a three-phase motor, a controller, a three-phase inverter, a host computer and more. The controlled motor is a permanent magnet synchronous motor with a power of 1.5 KW, and the motor parameters are illustrated in Table 1. The controller is based on the RTU-BOX204 real-time digital control platform. The magnetic powder brake is used to generate the load torque. The parameters of the ANTSM controller are chosen as  $\beta = 600$ ,  $q = 11$ ,  $p = 17$ ,  $\eta = 1.5$ ,  $\varepsilon = 0.99$ ,  $N = 80$ ,  $k_m = 1$  and  $k_M = 30$ , and the parameters of MESO are selected as  $h_1 = 30$  and  $h_2 = 225$ .



**Figure 8.** Response curve under the step change of speed in the start-up phase. (a)ANTSM+ESO. (b) ANTSM + MESO.

#### 4.1. Comparison of experimental results among PI, traditional NTSM and ANTSM

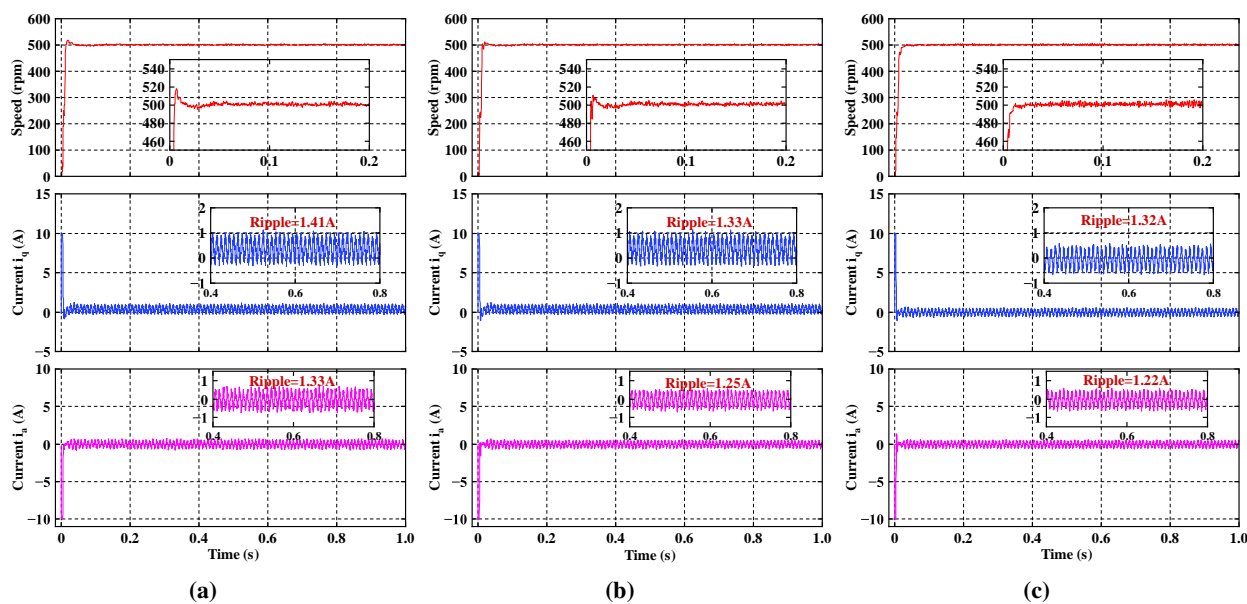
The start-up results of  $\omega$ ,  $i_q$  and  $i_a$  under the PI, traditional NTSM and the proposed ANTSM controller are respectively depicted in Figure 5. The given speed is 500 rpm, and no additional load torque was added. One can notice that the start-up transient response under the PI has the longest convergence time and largest speed overshoot. In Figure 5(c), the developed ANTSM controller has the smaller start speed overshoot. Figure 6 shows a sudden load experiment. From the experimental results, one can see that the proposed ANTSM controller has the smallest speed fluctuation and current ripple.

#### 4.2. Comparison of experimental results among ANTSM+ESO and ANTSM+MESO

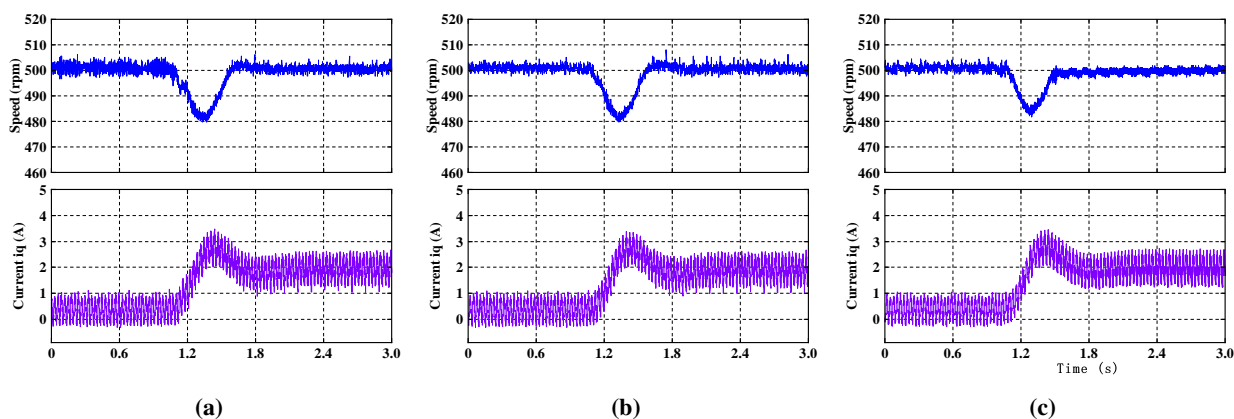
In this subsection, the ESO-based ANTSM controller is utilized to compare with the presented MESO-based ANTSM controller. Figure 7 exhibits the sudden load change responses at the speed of 500 rpm. After loading, the speed recovery time under the proposed ANTSM+MESO controller is shorter than that under the ANTSM+ESO controller, and the speed fluctuation under the proposed ANTSM+MESO controller is also smaller. The starting responses are depicted by Figure 8. One can undoubtedly observe that the speed overshoot of the proposed ANTSM+MESO controller is smaller than that under the ANTSM+ESO. Additionally, the time for the q-axis and a-phase currents to achieve the system stability was shorter than that under the ANTSM+ESO controller.

#### 4.3. Experimental results of dynamic performance tests

To further validate the availability of the proposed controller, numerous additional experiments were carried out. The moment of inertia has a straightforward impact on the starting and braking performance of the motor. As a result, the controller performance was tested by changing the value of



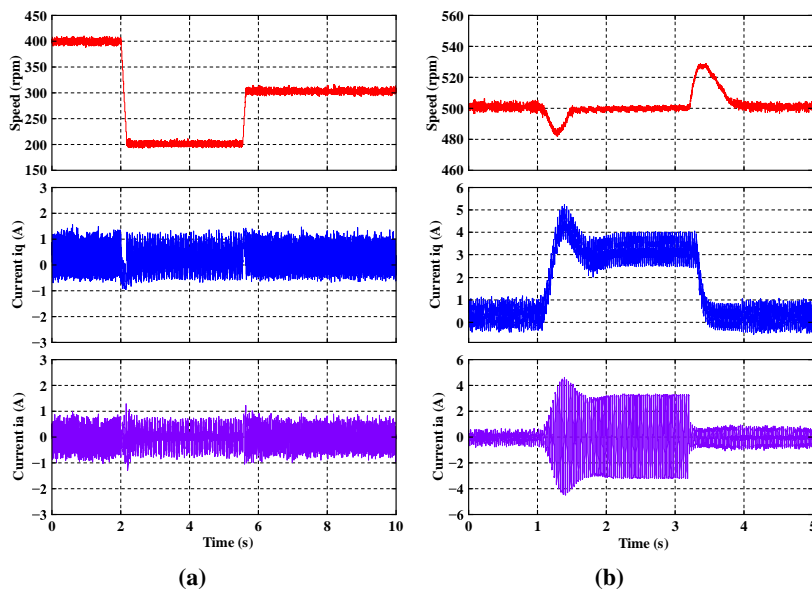
**Figure 9.** Step response of the ANTSM+MESO controller. (a)  $J = (1/3) J_0$ . (b)  $J = (1/2) J_0$ . (c)  $J = J_0$ .



**Figure 10.** Speed responses of the ANTSM+MESO controller under a sudden load torque change. (a)  $J = (1/3) J_0$ . (b)  $J = (1/2) J_0$ . (c)  $J = J_0$ .

inertia in the presented ANTSM+MESO controller. The inertia  $J = (1/3) J_0$ ,  $J = (1/2) J_0$  and  $J = J_0$  were chosen for comparative experiments. Figure 9 shows the step speed responses at the three different inertia. The sudden load change responses at the speed of 500 rpm are exhibited by Figure 10. As can be seen in Figures 9 and 10, the inertia mismatch imposes an impact on the stable operation of the motor. Nevertheless, the system can still operate stably under the proposed ANTSM+MESO controller.

The experimental results of speed, current  $i_q$  and current  $i_a$  during the speed change are shown in Figure 11(a). The anti-disturbance property of the ANTSM + MESO control strategy at 500 rpm is exhibited in Figure 11(b). From Figure 11, one can summarize that the ANTSM + MESO algorithm can better control the motor operation at a wide range of speed.



**Figure 11.** Response curves of ANTSM + MESO controller in low speed region.

## 5. Conclusions

In this paper, a novel MESO-based ANTSM controller was constructed to improve the anti-disturbance performance of the PMSM drive system with parameter variation and unknown disturbance. To prevent the unsatisfactory control effect due to overestimating switching gain, an adaptive law was combined with the traditional NTSM strategy to achieve the expected performance. Furthermore, compensating the disturbance estimation value obtained by MESO to the ANTSM controller, the switching gain can be further reduced. Comprehensive experimental results demonstrate that the property of the MESO-based ANTSM algorithm outperforms both traditional PI and NTSM control methods. In future work, we will work on solving the stability problem caused by parameter variations.

## Use of AI tools declaration

The authors declare they have not used Artificial Intelligence (AI) tools in the creation of this article.

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## Conflict of interest

The authors declare there is no conflict of interest.

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